

TEST OF A 400 H. P. GAS PRODUCER  
OF  
ARMOUR & CO. CHICAGO, ILLINOIS

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ARMOUR INSTITUTE OF TECHNOLOGY

1909

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producer at Armour & Co.,







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TEST  
OF  
A 400 H. P. GAS PRODUCER  
AT  
ARMOUR & CO., CHICAGO, ILL.

A THESIS

PRESENTED BY

K. M. BOBLETT

&

N. J. BOUGHTON

TO THE

PRESIDENT AND FACULTY

OF

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FOR THE DEGREE OF

BACHELOR OF SCIENCE IN MECHANICAL ENGINEERING

HAVING COMPLETED THE PRESCRIBED COURSE OF STUDY IN

MECHANICAL ENGINEERING

1909

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## Preface.

The subject of " A Test of a 400 H.P. Gas Producer" is herein presented in five parts.

Part 1:- A brief outline of the underlying principles of a gas producer and the reactions which bring about the change from solid to gaseous fuel.

Part 2:- A description of the plant and the apparatus used in making the test.

Part 3:- The actual test of this particular plant together with the data obtained, and the results.

Part 4:- A brief discussion of the various items which enter into the cost of generating gas in a gas producer, and the total cost of this particular gas.

Part 5:- Conclusions and Bibliography.



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## Test Of A 400 H.P. Gas Producer.

### Part 1.

A brief outline of the underlying principles of a gas producer and the reactions which bring about the change from solid to gas~~eous~~ fuel.



## Test Of A 400 H.P. Gas Producer.

The question of fuel is such a large factor in the maintenance of industrial prosperity that there is an increasing demand for closer approximation in daily practice to theoretical efficiency.

There are three classes of fuel; coal, which is the most plentiful; oil, which has the maximum energy; and gas, which can be used in the most ideal manner. Natural gas has been a boon, and at the same time a drawback to producer gas, in so far that the discovery of natural gas and its application to industrial purposes has been the means of teaching the public the many and various purposes to which the gas can be put. Again in the natural gas belts this gas is so cheap that man's gas producers cannot compete with nature's producers.

Any fuel, other than natural gas, must be first gasified before heat will be evolved. Whether the resulting gases are combustible, or not, depends upon the nature of the fuel and the means of gasification. The products of complete combustion of ordinary fuels are, carbon dioxide ( $\text{CO}_2$ ) and water ( $\text{H}_2\text{O}$ ).



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The object of direct firing is to effect vaporization, distillation, gasification and complete combustion in one chamber and in close proximity to the fuel bed. Heat is required to vaporize, distill and gasify a fuel, while heat is given up during the combustion of the gases.

The underlying principle of a gas producer is to separate as much as possible, the gasification and the complete combustion of the fuel, for the high temperature resulting from the combustion of the gas may come about in a different locality to that of the first processes of the heat change.

The following reasons are then apparent as to the advantages of gas producers:-

- First - More complete combustion.
- Second - Easy control of the air and gas supply, and consequently the combustion.
- Third - Higher possible temperatures.
- Fourth - Less heat lost up the chimney.
- Fifth - No loss of fuel thru the grate.
- Sixth - No irregularities of firing.
- Seventh - Greater efficiency in the transfer of



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heat, due to the higher temperature.

Eighth - It concentrates and lessens labor in handling coal and ash; eliminates the deleterious effects of the ash upon the substance heated and reduces the smoke nuisance to a minimum.

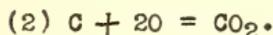
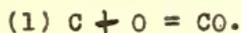
A gas producer, as the name implies, is an apparatus which generates a combustible gas by the incomplete combustion of a solid fuel. Industrially, producer gas is the combustible product of a rather complex series of physical and chemical changes induced in the fuel by the heat arising from its complete combustion. The combustion is termed incomplete not that it leaves part of the fuel unburnt as a residue of carbon or coke, but as being so unburnt that, while completely gasified, the fuel only generates about 30% of its heat, primary combustion, in the producer.

In all cases, carbon is the basic element of the fuel. The heat necessary for the producer reactions and the quantitative yield of gas is dependent upon its amount and degree of oxidation. As the chief duty of the gas producer is

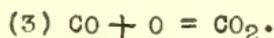


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to gasify the solid carbonaceous constituent of the fuel, this must become the first consideration. Carbon has the property of combining with oxygen in two ways, thus:



In the first combination 12 parts by weight of carbon combine with 16 parts by weight of oxygen to form 28 parts by weight of carbon monoxide (CO). During this partial oxidation of the carbon, 4451 B.T.U. is developed, for each pound of carbon. The CO thus formed is capable of developing 10159 B.T.U. on the complete oxidation of the carbon according to the equation,



Under normal conditions of temperature, carbon and oxygen have practically no action upon each other, so that, to bring about combustion, heat must be applied to one or the other, or both, of the constituents. The bringing about of this chemical combination by the application of heat is more dependent upon the intensity of the heat than the quantity, as when the combustion has been



### Test Of A 400 H.P. Gas Producer.

brought about, in however small a degree, the heat developed by the combustion causes the continuation of the combination until one or the other of the constituents is consumed.

Since the air, which usually furnishes the oxygen for combustion contains 3.77 volumes of nitrogen for every volume of oxygen, it is desirable to get at least a part of the oxygen from a source free from nitrogen. Steam is used to furnish this additional oxygen and at the same time enrich the gas by the free hydrogen. Since the breaking up of the steam is a heat absorbing process, its introduction into a gas producer has the property of cooling the fuel column and thus lessens the formation of clinkers.

Assuming at first that all the oxygen for combustion is furnished by the air, then there is in the gaseous mixture,

3.77 volumes of nitrogen = 1349 cu.ft. or 65.3%.  
 2.00 volumes of carbon monoxide = 715 cu.ft. or 34.7%.

Or 5.77 volumes of the gas = 2064 cu.ft. from 24 pounds of carbon. This follows from the eq-



## Test Of A 400 H.P. Gas Producer.

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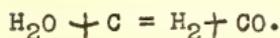
$2C + O_2 = 2CO$ . That is, 2 volumes of carbon combine with one volume of oxygen to form 2 volumes of carbon monoxide. Also 24 pounds of carbon combine with 32 pounds of oxygen to form 56 pounds of carbon monoxide. Therefore 1 lb. C produces (  $2064 + 24$  ) 86 cu.ft. of gas. This 56 lbs. CO is capable of generating (  $1059 \times 56$  ) 243815 B.T.U. or the gas will have a heat value of (  $243815 + 2064$  ) 118.1 B.T.U. per cu.ft., and the CO contains (  $243815 + 715$  ) 341 B.T.U. per cu.ft.

If 10% of the sensible heat in the gas is allowed for radiation, conduction and the heating of the fuel, there still remains (  $2/3 \ 4451$  ) 2967 B.T.U. per pound of carbon, as sensible heat. This heat will raise the temperature of the gas (  $2967 + 86 \times .019$  ) 1816 Deg.F. The temperature, however, of the fuel bed will be much higher than this and if some agent were introduced to absorb a part of this heat of primary combustion, the temperature will be lowered and clinkering will be minimized. For the purpose of absorbing a part of the heat of primary combustion a mixture of steam and air is used instead of air alone.



### Test Of A 400 H.P. Gas Producer.

As the heat transfer in the fuel column is so complex as to prohibit the actual calculation of the theoretical amount of steam used, it will be assumed that 20% of the carbon is oxidized by the decomposition of the steam while 80% is oxidized by the oxygen from the air. It will be assumed that part of the heat lost by radiation has gone to raise the temperature of the water to 212 Deg.F. and evaporate it. Then from the equation,

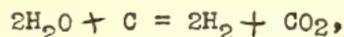


That is, 18 lbs. of water is decomposed by 12 lbs. carbon, to form 2 lbs. hydrogen and 28 lbs. carbon monoxide. The heat absorbed in decomposing 18 lbs. steam at 212 Deg.F. is  $(52290 \times 2)$  104580 B.T.U. Heat developed by the partial oxidation of the carbon =  $(4451 \times 12)$  53412 B.T.U.. The net efficiency then =  $(104580 - 53412)$  51168 B.T.U., or per pound of steam the heat deficiency is  $(51168 \div 18)$  2843 B.T.U.. The heat produced per pound of carbon burnt to CO by the oxygen of the air (allowing for radiation, etc.) is 2967 B.T.U.. Therefore the maximum amount of steam which may



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be decomposed per pound of carbon burnt by the air is  $(2843 + 2967) \cdot 96$  lbs. , In practice, however, some of the carbon which is completely oxidized in the lower part of the fuel column, to  $\text{CO}_2$ , finds its way thru the upper part of the fuel column and into the gas without being reduced to  $\text{CO}$ , and by so doing, more heat is available to break up steam and make up for the heat deficiency. From the equation,



we see that the decomposition of 36 lbs. of steam will absorb  $(52290 \times 4) 209160$  B.T.U., and the heat developed by the oxidation of the carbon by the oxygen of the steam is  $(14648 \times 12) 175776$  B.T.U., and the net heat absorption is  $209160 - 175776 = 33384$  B.T.U., or per pound of steam is  $(33384 + 36) 928$  B.T.U. Therefore the maximum amount of steam which may be decomposed per pound of carbon burnt by the air is  $(2967 + 928) 3.2$  lbs.

Since, in the process where no  $\text{CO}_2$  is formed, one-third of the total carbon is oxidized by the steam, it follows that the maximum amount of steam which may be decomposed by the oxidation of the



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carbon with air and steam is  $(.96 + 1.5) .64$  lbs. steam; and where the  $CO_2$  is formed the maximum amount of steam is  $(1.49 + 3.2) 2.15$  lbs. steam.

( In the first case the steam oxidizes two-thirds, and in the second case one-third of its weight in carbon).

The different reactions may thus be classified:-

- 1st. No steam used and all the carbon burnt to CO.
- 2nd. Steam used and all the carbon burnt to CO.
- 3d. Steam used and all the carbon burnt to  $CO_2$ .

In the second reaction the steam oxidizes two-thirds, and in the third, one-third its weight in carbon. Table 1 shows the quality and quantity of the gas by the various combinations of the different reactions.

It is obvious that where the gas of highest heat value is desired, the reactions should follow that of No. 2. As this reaction requires a furnace temperature of about 1900 Deg. F., it is hardly practicable to use it, so a combination of reactions No. 2, and No. 3 is used. However there is a limit to which steam may be used, as the excessive for-



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TABLE 1

CONSTITUENTS OF THE GAS	REACTION NO. 1		REACTIONS NOS. 1 & 2 COMBINED		REACTIONS NOS. 1 & 3 COMBINED	
	VOLUMES CU. FT.	% BY VOL.	VOLUMES CU. FT.	% BY VOL.	VOLUMES CU. FT.	% BY VOL.
CO FROM AIR	2.00	34.7	2.00	715.6	2.00	715.6
CO FROM STEAM	---	---	1.00	357.0	---	---
H FROM STEAM	---	---	1.00	367.8	1.96	701.2
CO <sub>2</sub> FROM STEAM	---	---	---	---	0.90	350.6
N FROM AIR	3.77	65.3	3.77	1348.8	3.77	1348.8
VOLUME OF GAS PRODUCED	5.77	100.	77.6	2780.0	8.70	3116.2
VOLUME OF GAS PER LB. CARBON	86 CUBIC FEET		772 CUBIC FEET		872 CUBIC FEET	
COMBUSTIBLE IN GAS	34.7 %		51.5 %		45.5 %	
CALORIFIC POWER PER CUBIC FOOT	119 B.T.U.		176.9 B.T.U.		156.8 B.T.U.	
STEAM DECOMPOSED PER LB. OF CARBON	NONE USED		.64 LB.		2.15 LB.	
AIR REQUIRED PER LB. CARBON	712 CUBIC FEET.		475 CUBIC FEET.		47.8 CUBIC FEET.	

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GASES BY VOLUME		EXCESS OF STEAM.		
		MODERATE	GREAT	MAXIMUM.
CO <sub>2</sub>	-----	5.30 %	8.90 %	15.00 %
CO	-----	23.50	16.40	11.50
CH <sub>4</sub>	-----	3.30	2.55	1.90
H	-----	13.14	18.60	24.60
HEAT VALUE PER CU.F.T.		151 B.T.U.	135 B.T.U.	129 B.T.U.
TEMPERATURE.		1472°F.	1292°F.	932°F.

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mation of  $\text{CO}_2$  greatly lowers the heating value of the gas from the lowering of the furnace temperature. Table 2 shows the effect on the gas of different amounts of steam.

The use of the exhaust gases from gas engines have been used as a cooling agent, but the large per cent of inert nitrogen prohibits its use, for the resulting gas will have a low heat value owing to the large per cent of nitrogen present.

The object of a gas producer is not only to gasify solid fuel, but to wash and cool the gas so that it may be easier transported or used directly in an internal combustion engine. Fig. 1-A is an elevation in section of a typical installation. Referring to Fig. 1-A, the gas is generated in the receptacle A, which consists of either a cast iron or sheet steel shell lined with some refractory earth as firebrick. The air and steam are lead in to the bottom of the generator thru a tuyere located in the center of the generator. This tuyere delivers the moisture laden air a few inches below the top of the ash bed, where it has a chance to travel up thru the fuel bed un-





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iformly. Coal is fed into the hopper on top of the producer, and this hopper is so constructed that when it is open at the top to receive coal, it is closed at the bottom to prevent air from leaking in, if it is a suction producer; or the gas leaking out, if it is a pressure producer.

As the gas comes from the fuel bed in the generator it is led to the economizer B. This economizer consists of two concentric steel shells M, & M', so arranged that the gases pass down thru the inner shell while the air for combustion ascends within the annular space between the shells, passing out at the top as shown. Steam is supplied to the air by the pipe P. The steam meets the air at the throat of the venturi tube V, where, in passing downward and expanding, causes the air to be drawn with it. This steam jet causes a slight pressure in the bottom of the generator, and should be just enough to overcome the resistance of the fuel column. The air enters the economizer thru a slide shutter at D. From the economizer the gas passes to the scrubber and in doing so, its course is changed at T which is a short pipe,



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water sealed at S. The dust and solid particles in the gas tend to go in the same direction as before, and most of this solid matter is deposited in the water seal S where it is removed. The scrubber consists of a steel tank. The space .. is filled with coke down thru which water from I trickles. The water meets the upflow of gas and cools it as well as removing the remaining dust and solid matter. The coke is for the purpose of presenting a large wetted surface in contact with the gas. The space E is filled with excelsior soaked with oil. This excelsior removes all the entrained moisture in the gas so that when the gas leaves the scrubber it is cool, but saturated with aqueous vapor at the temperature of the gas. The scrubber water leaves at some point at or around the bottom of the scrubber, as at I'.

When steam is not furnished from some outside source the generator is water jacketed as is the top and the steam thus formed is drawn in with the air. With a producer of this type, instead of there being a pressure in the bottom



### Test Of A 400 H.P. Gas Producer.

of the generator there is a slight suction, and the exhauster, or engine, must draw the gas thru the entire system. A disadvantage of this method lies in the fact that when the operator pokes the fuel bed, air leaks in, not only burning the gas, but diluting it with air which lowers its heating value.

In the case of the producer tested the pressure in the top of the generator is maintained at or near atmospheric pressure, so that the fuel bed could be put in order without in any way lowering the quality of the gas.

#### Requirements of a producer installation:-

A producer, to be a good one, must be reliable and make a uniform gas under all conditions of load. It must also be able to meet as much as possible, the fluctuations of the load. The producer should be such that it can be quickly started and simple to operate, since the wages paid to operators of gas producers do not induce<sup>d</sup> educated and skilled mechanics to enter this field.

A gas producer should be able to hold fire for at least seventy hours, as from Saturday at noon to Monday morning at six o'clock, without



### Test Of A 400 H.P. Gas Producer.

any attendance and without much fuel being used. After standing for this length of time the producer should be such that good gas can be made twenty-five minutes after starting up.

The one prime factor in the operation of a producer is that it must be economical of fuel. This means that the loss thru the ash-pit must be small, also a large per cent of the heat which would otherwise be wasted should be returned to the fuel bed by preheating the air or evaporating water.

A producer should be so designed that under normal working conditions clinkers will not form. Too high a rate of gassing in a producer, and an insufficient supply of steam will cause clinkers since with American fuels a temperature above 1500 Deg.F. in the generator will cause clinkers. If coal could be obtained in this country free from a fusible ash, producers could be run at a higher temperature, and consequently at a higher efficiency.



Test Of A 400 H.P. Gas Producer.

Part 2.

A description of the plant, and the apparatus used in making the test.



## Test Of A 400 H.P. Gas Producer.

Location of the plant:- The producer gas plant which was tested is installed at the works of Armour & Co., Union Stock Yards, and furnishes gas in their cooking, soldering, and canning departments. Figs. 1, 2, & 3, show the general layout of the plant as was installed by Flinn & Drefflein, Engineers, Chicago, Ill. The plant is rated as a 400 H.P. generating unit; that is, the plant is capable of supplying gas to a 400 H.P. producer gas engine, which has a thermal efficiency of 20%.

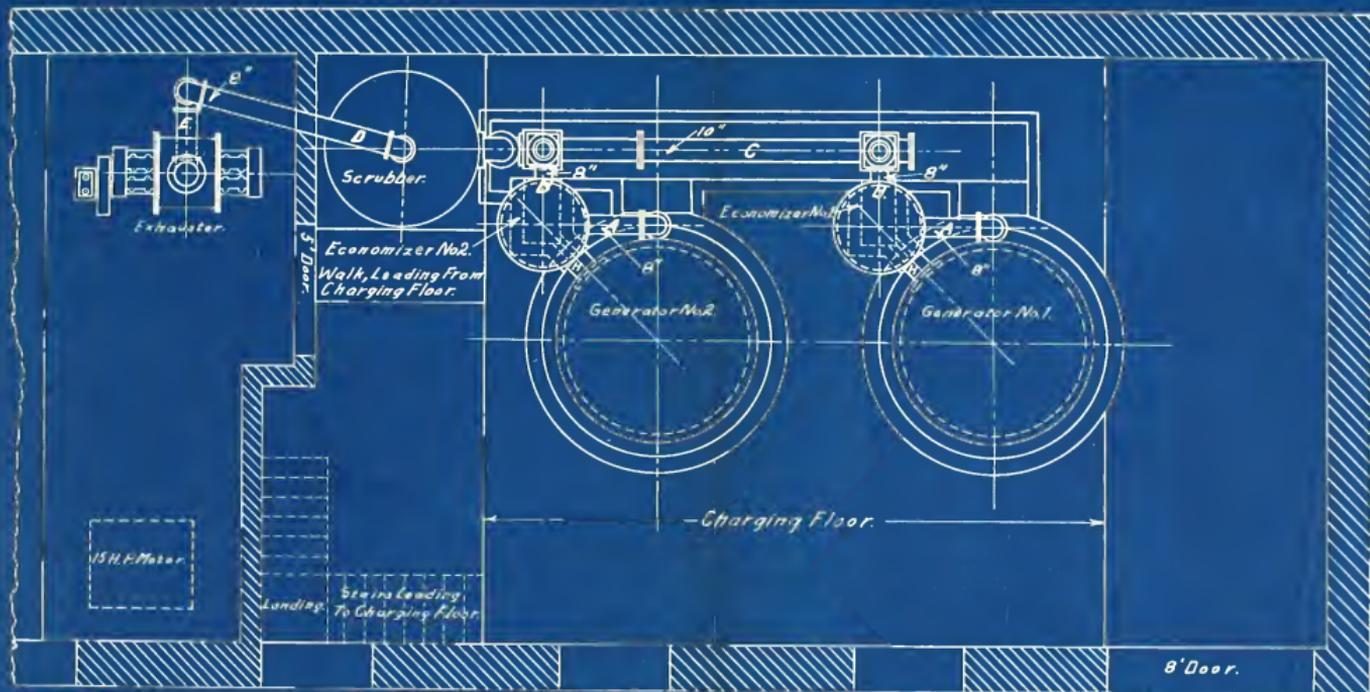
Operation of the plant:- A producer consists of three essential parts; a generator, in which the coal is gasified, the economizer, which preheats the air, and the scrubber, which cools the gas and at the same time cleanses it.

The coal is elevated to the coal bunkers by the coal conveyer, as shown in Fig. 2. From the bunkers the coal gravitates into the weighing hoppers, one over each generator, where it is weighed. From the weighing hoppers it goes to the charging hoppers, which are so arranged that when the hoppers are open at the top they are closed at the bottom, preventing the escape of the gas while



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Unloading Platform, 7' Wide.

Fig. 1.

PLAN  
 OF  
 PRODUCER GAS PLANT  
 AT  
 ARMOUR & CO.,  
 UNION STOCK YARDS,

CHICAGO, ILLS.  
 JAN. 18th, 1902.

SCALE,  $\frac{1}{4}'' = 1'-0''$ .

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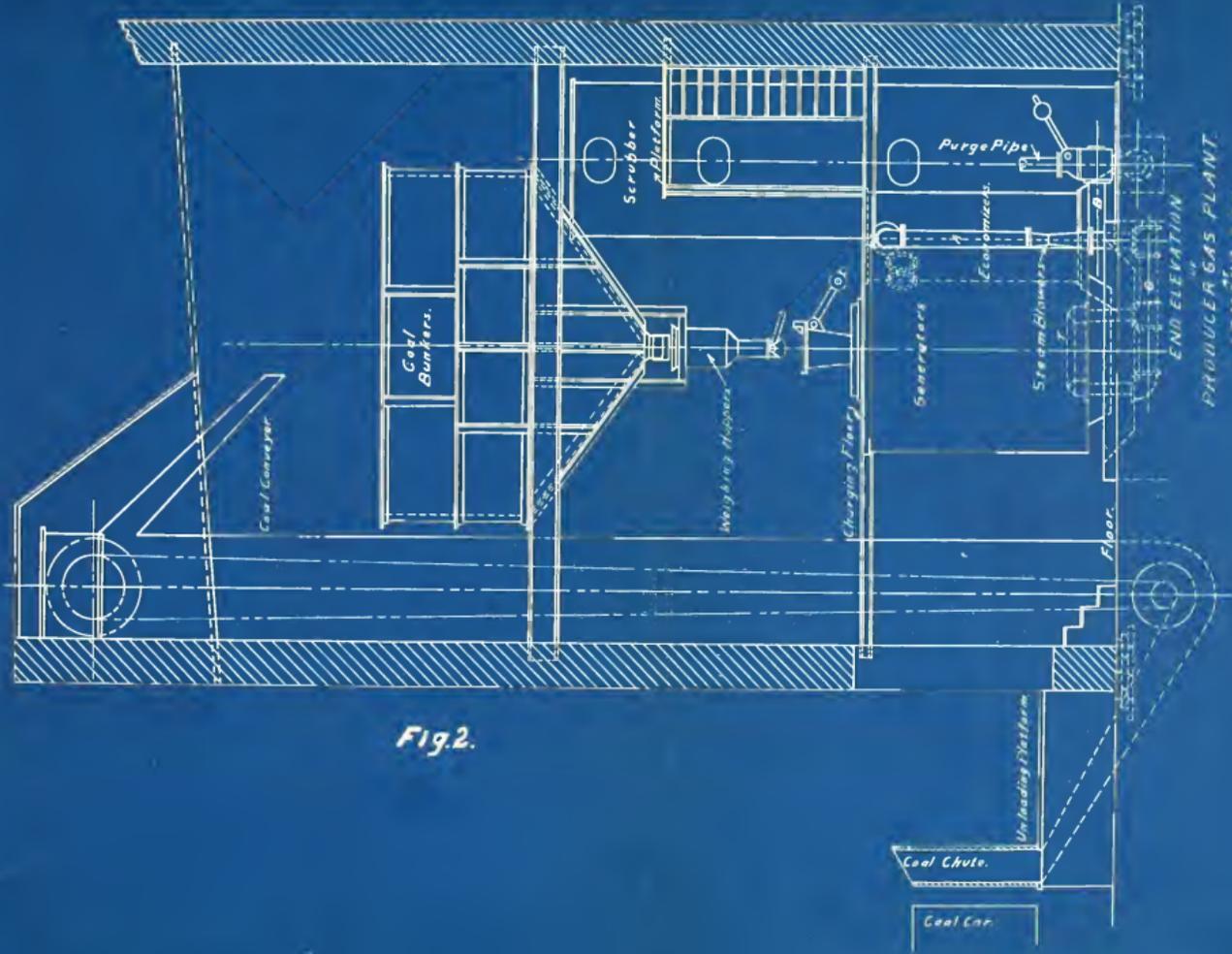


Fig. 2.

PRODUCER GAS PLANT.  
 ARMOUR & CO.  
 UNION STOCK YARDS.  
 CHICAGO, ILLS.  
 JAN. 1898.  
 SCALE 3/4"=1'-0"

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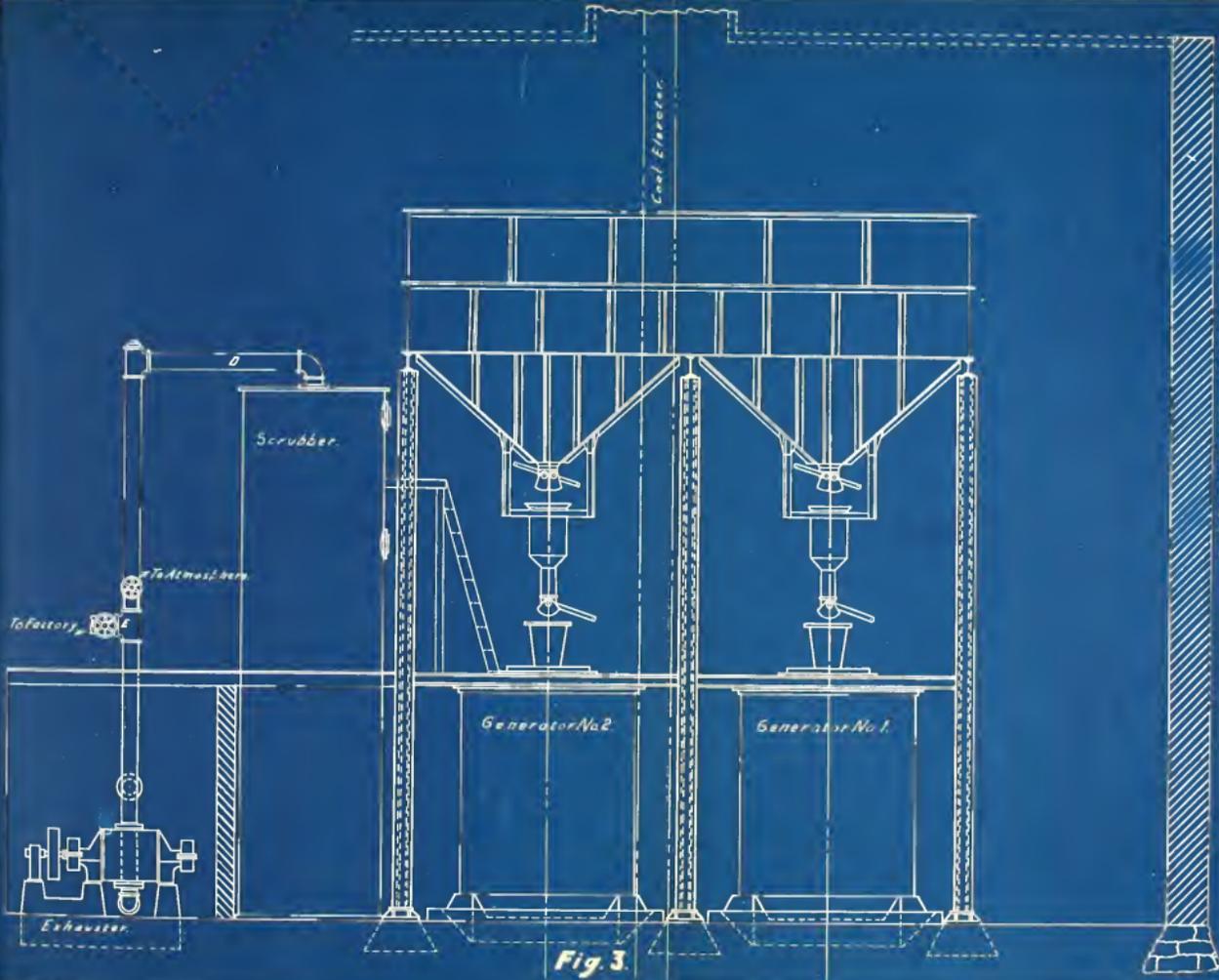


Fig. 3.

SIDE ELEVATION OF  
 PRODUCER GAS PLANT  
 UNION ARMOUR & CO.  
 UNION STOCK YARDS, CHICAGO, ILL.  
 Scale 1/4" = 1'-0"

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charging the noppers with coal. Fires are started in the generators by first filling the producers with ashes so that they just cover the circular tuyere irons (Fig.2). Upon this bed of ashes a wood fire is kindled, the supply of air being furnished by an auxiliary pressure blower. In starting up, the gas valve to the factory mains is closed and the purge pipe valve is opened to the atmosphere. When the wood fire is sufficiently under way, coal is added from the charging hopper until there is an incandescent bed of fuel four to five feet deep. When it is supposed that the gas is of sufficient quality it is tested by lighting a burner, and if found so, the exhaustor is started, the factory gas valve opened, the purge pipe valves closed, steam turned into the air, and the producer is started for an indefinite run.

Steam is furnished to the generator from the company's steam power plant at about 140 lbs. pressure which is throttled down to suit the requirements. The steam, on passing thru the venturimeter is expanded down to a few inches of water pressure. This expansion of the steam creates a suction of air thru the economizer and at the



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same time supersaturates the air with moisture.

The generators are two in number, are 13'-0" high above the floor, are 7'-6" external diameter and 6'-0" internal diameter. They are each set in a concrete basin and rest upon four concrete pillars. The bottom of the shell is about 8" from the bottom of the basin, which allows the ashes to be removed without disturbing the gas making process. This concrete basin is about 12" deep and is kept filled with water at all times. The generators are lined with fire brick on the sides and on the top, with the exception of the pose holes and the charging hopper. To prevent undue heating of the hopper casting, the top is water cooled, the overflow being allowed to gravitate to the basin below. The gas leaves the generator thru the 10" pipes H (Fig.1), and enters the economizers, leaving by the 10" pipes B. The gas, on passing thru the economizers, preheats the air.

The economizers consist of two concentric cylindrical shells, with a narrow air space between them. The gas passes down thru the inner shell while the air ascends thru the annular space be-



## Test Of A 400 H.P. Gas Producer.

tween the shells, causing the economizers to operate on the counter-current principle. The preheated air passes thru the 8" pipe A, down thru the steam blower, thru the pipe G (Fig.2) and is finally distributed thru the fuel bed by the tuyere irons T (Fig.2).

The gas passes on its way to the scrubber from the pipes B to the 10" main C, enters the bottom of the scrubber, and leaves from the top thru the 8" pipe D. The exhauster (A Roots positive blower) delivers the gas to the factory mains E under a pressure of about 2.5 lbs. Fig. 2-A shows the general arrangement of the piping to the auxiliary exhauster, and to the main exhauster.

Water enters the scrubber at the top, percolates thru a bed of coke, discharging to the sewer at the bottom. The gas, ascending thru this bed of coke meets the downward flow of water, and is thereby cooled and cleansed. Excelsior in the top compartment of the scrubber removes the entrained moisture, so that the gas is as free from moisture as is possible to make it under ordinary conditions



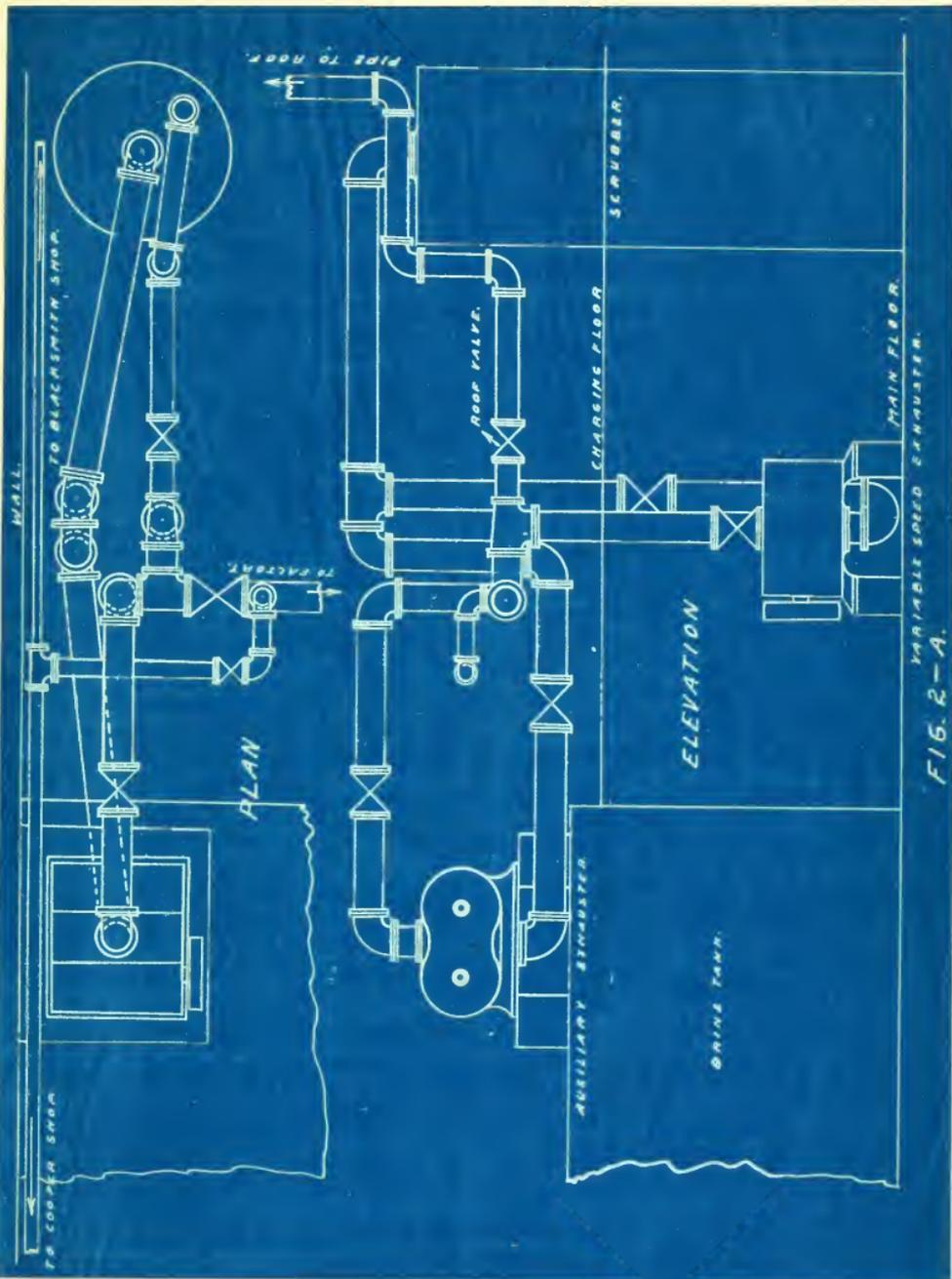


FIG. 2-A.

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### Test Of A 400 H.P. Gas Producer.

and is as dry as need be for ordinary purposes.

Arrangement of the apparatus for the test:-

To obtain the total water used in the plant, a 1 1/2" Empire water meter was placed in the water supply line and by-passed so that it could be cut out of service when not being used for testing. The water used by the scrubber was measured by a 1 1/2" Keystone water meter placed close to the scrubber, by-passed so that it could also be cut out of service when not in use for test purposes. These meters were thus arranged on account of the solid matter in the water, which would soon foul the meters and so cause them to give incorrect results. The difference between the readings of the two meters gave the amount of water used in the water cooled top of the generators and in the ash-pit seal.

Temperatures:- The temperature of the generator and the gas leaving the generator was obtained with a Hoskins Thermo-electric couple used in connection with a Weston milli voltmeter. The combination was calibrated against a Callander Electric Resistance Pyrometer before and after the test.



### Test Of A 400 H.P. Gas Producer.

The pyrometer was inserted in one of the poke holes, and fire clay placed around it to prevent the escape of gas.

All other temperatures measured in this test were taken with mercurial thermometers, placed in the following positions; the temperature of the outside air was obtained by placing a 0-220 Deg.F. thermometer outside the building; temperature of the room by placing a 0-220 Deg.F. thermometer at some convenient place in the producer room; temperature of the gas from the economizer, by placing a 0-800 Deg.F. thermometer in the pipe B (Fig.2); temperature of the gas to the scrubber, by placing a 0-600 Deg.F. thermometer in the gas main just where it enters the scrubber; temperature of the gas leaving the scrubber, by placing a 0-220 Deg.F. thermometer in the pipe D (Fig.1); temperature of the gas in the factory mains, by placing a 0-220 Deg.F. thermometer in the pipe leading from the exhauster; temperature of the air leaving the economizer, by placing a 0-600 Deg. F. thermometer in the pipe A (Fig.1).

The humidity of the atmosphere was obtained

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## Test Of A 400 H.P. Gas Producer.

by placing a wet and dry bulb thermometer in the room.

Pressures:- The pressure of the gas in the mains was obtained from a mercurial pressure gauge, graduated to read in pounds and ounces. The pressure of the steam was read from a Bourdon steam gauge, calibrated and found correct. U-tube water gauges were used to obtain all other pressures except the barometer observations.

The power consumed by the motor running the exhauster was measured by an integrating watt-meter. An attempt was made to measure the gas with a Pitot tube placed in the pipe leading from the exhauster, but the pulsating flow of the gas prevented any readings being taken.

Analyses of the coal and ash were obtained in the standard manner of quartering. The gas analysis was made with a Morehead Burette. The heating value of the gas was obtained by burning it in a Junier Gas calorimeter.

The humidity of the gas was obtained by placing a calcium chloride tube (Fig.4-A) upon the discharge pipe from the exhauster and a known a-

The first part of the book is devoted to a general history of the United States from its discovery to the present time. It is divided into three volumes, the first of which contains the history of the discovery and settlement of the continent, the second the history of the colonies, and the third the history of the United States since its independence.

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## Test Of A 400 H.P. Gas Producer.

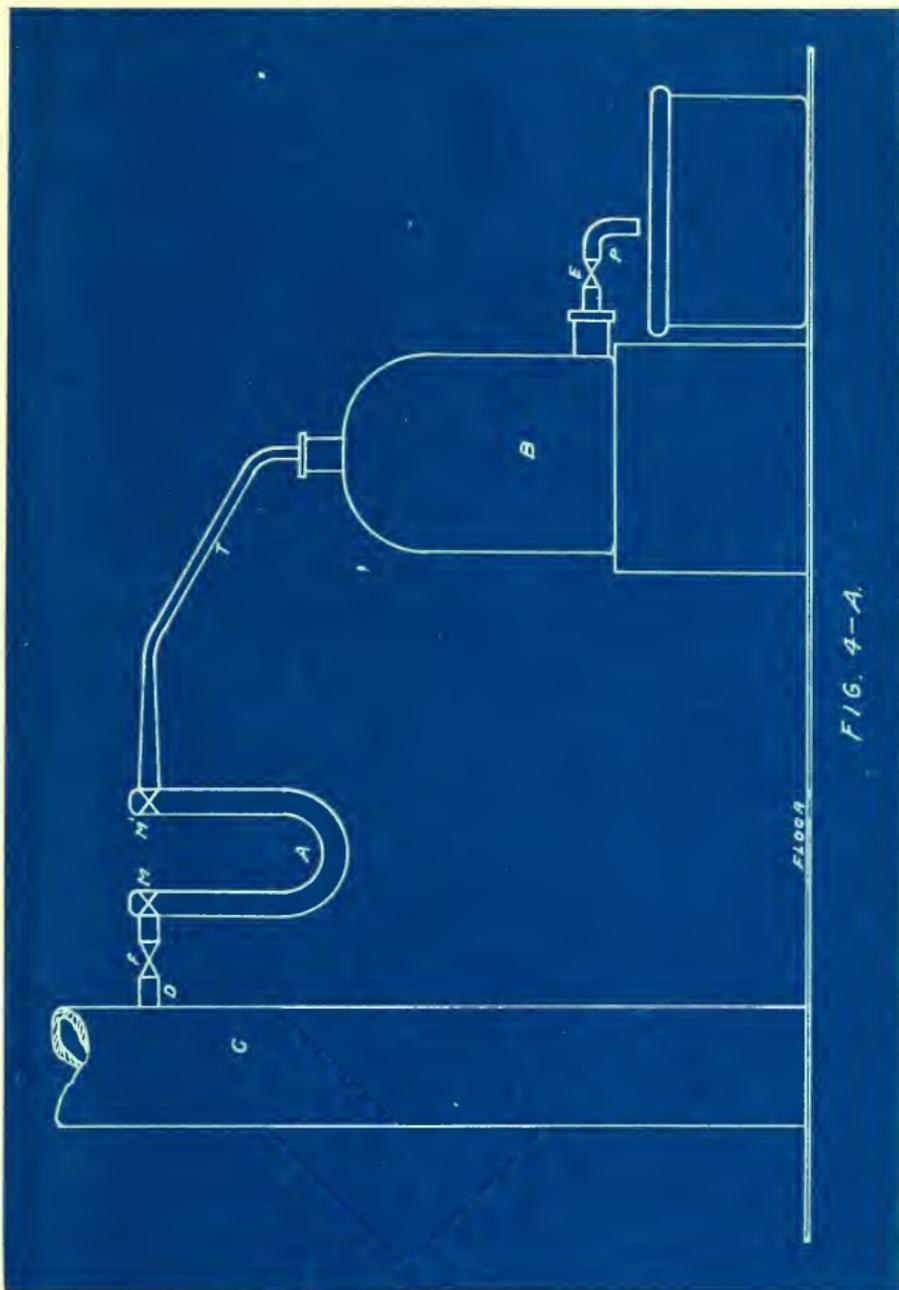


FIG. 4-A.

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### Test Of A 400 H.P. Gas Producer.

mount of gas thru. Referring to the figure, the gas main C was tapped and the pipe D with valve E placed in it. A rubber tube connected the U-tube to the pipe D. These connections were made as short as possible. The other leg of the U-tube was connected to the water bottle B by the rubber tube T. The ground glass cocks M, M' serve to regulate the flow of the gas. The water is allowed to drain from the bottle by means of the connection E and valve P.

In operation, the bottle B is filled with water as is the tube T. After blowing the gas to the atmosphere from D, to remove the air, the connection at M is made, the valve P opened, M, & M' opened so as to allow about 100 C.C. of gas per minute to flow thru the U-tube. The amount of water which is displaced (run out from the connection E) is caught and measured and represents the amount of gas passed thru the U-tube. By weighing the U-tube before and after using it the amount of water given up by the gas is found. If  $w$  = the weight in grams of the moisture absorbed, then  $w \div 453.6$  = the weight in pounds of the moisture



### Test Of A 400 H.P. Gas Producer.

absorbed.

The cubic feet of gas passed thru = .0000353 M,  
where M is the C.C. of water displaced by the gas.

Then the pounds of water per cubic foot of  
gas =  $W = w + (.453.6 \times .0000353) M = w + .0162 M$ .

Knowing from Table 58, Gebhardt's Steam Power  
Plant Engineering, the weight of moisture in one  
cubic foot of the gas at the temperature of the  
gas which will saturate the gas, the relative  
humidity of the gas in per cent is calculated  
from the ratio of the amount of moisture actually  
present in the gas, to the maximum saturation at  
that temperature.

#### General Dimensions of the Plant.

Diameter of the economizers, .....	3'-5".
Diameter of generators (internal), .	6'-0".
Diameter of scrubber, .....	6'-0".
height of economizers, .....	8'-7".
Height of generators, .....	10'-3".
Height of scrubber, .....	21'-9".
H.P. of exhauster motor, .....	25.
Kind of coal used, .....	Anthracite Pea.
Number of coal burners, .....	2.

CHAPTER I

THE EARLY HISTORY OF THE UNITED STATES

THE first discovery of the continent of North America was made by Christopher Columbus in 1492. He sailed from Spain in search of a westward route to the Indies, and after a long and perilous voyage, he landed on the island of San Salvador in the West Indies. This discovery opened up a new world of opportunity for the European powers, and led to the colonization of the continent.

The first permanent European settlement in North America was founded by John Rolfe in 1607. He established the Jamestown colony in Virginia, which became the first English colony in the New World. The colony was founded as a trading post for the Virginia Company, and its success depended on the cultivation of tobacco as a cash crop.

The Pilgrims, a group of English separatists, founded the Plymouth colony in Massachusetts in 1620. They sought religious freedom and a better life in the New World. The Pilgrims' experience at Plymouth is recorded in the book "The Pilgrim's Progress" by John Bunyan.

The Dutch established the first permanent settlement in the Netherlands, New Amsterdam, in 1614. This settlement later became the city of New York. The Dutch also established a colony in the Carolinas, which was later taken over by the British.

The French established a colony in the Mississippi Valley in 1682. This colony, known as Louisiana, was named after King Louis XIV of France. The French also established a colony in the Great Lakes region, which was later taken over by the British.

The Spanish established a colony in the Southwest in 1598. This colony, known as New Mexico, was founded by Juan de Oñate. The Spanish also established a colony in the Florida region, which was later taken over by the British.

The British established a colony in the Carolinas in 1733. This colony, known as the Georgia colony, was founded by James Oglethorpe. The Georgia colony was established as a refuge for debtors and as a buffer between the other colonies and Spanish Florida.

The American Revolution began in 1775, and led to the independence of the United States in 1776. The revolution was fought between the thirteen original colonies and the British Empire. The American Revolution was a significant event in the history of the United States, as it established the first independent nation in the Western Hemisphere.

Test of A 400 H.P. Gas Producer.

Number of economizers, ..... 2.  
 Number of purge pipes, ..... 2.  
 Number of scrubbers, ..... 1.  
 Number of generators, ..... 2.  
 Size of charging floor, ..... 22'-6" x 23'-10".  
 Size of exhauster room, ..... 13'-8" x 22'-6".  
 Size of generator room, ..... 22'-6" x 40'-0".  
 Source of power, ..... The central station.  
 Source of the steam, .... The central station.  
 Source of the water supply, ..... City mains.  
 Total capacity of the bunkers, ..... 90 tons.  
 Type of building, ..... Fire proof.  
 Type of exhauster, ..... Roots Blower.

No.	Description	Amount	Total
1	...	...	...
2	...	...	...
3	...	...	...
4	...	...	...
5	...	...	...
6	...	...	...
7	...	...	...
8	...	...	...
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## Test Of A 400 H.P. Gas Producer.

### Part 3.

The actual test of this particular plant together with the data obtained, calculations and the results.



## Test Of A 400 H.P. Gas Producer.

In making this test there was no chance to vary the load on the producer, as would be desirable in a test of this kind. The producer had to be used to generate gas for factory use just as was required, and under these conditions the tests were run. To get as great a variation as possible, and to get comparable results, it was decided to run tests on eight separate days so that by chance there would be some variation in the load. The enclosed data shows that there was a very marked variation from very light load to about five-eighths load. Only one generator was tested at a time since the load was such that one generator could easily carry it.

The alternate method of firing was used, since in a test of this nature there would be a great deal of uncertainty in starting this size producer with green fires, and conditions<sup>s</sup> would hardly be adjusted in the time allotted to the test.

An attempt was made to meter the gas with a Pitot Tube, but, as before explained, the pulsating character of the exhauster disturbed the readings of the draft gauge in such a way that



### Test Of A 400 H.P. Gas Producer.

no reliable results could be obtained. Also the sulphur in the gas soon attacked the material of which the tube was made and within about six hours the velocity tube was clogged. The air was not metered with a Pitot tube, because we found that it would be impossible to calibrate the instrument at the temperature which it would be used.

With these difficulties not overcome, it was decided to calculate the amounts of gas, air, steam and refuse, as outlined in the succeeding pages. The following table gives the factors used in all of the calculations. All factors are referred to 30" mercury and 62 Deg. F.

Items	Air	CO <sub>2</sub>	CO	H	CH <sub>4</sub>	N
Wt./cu.ft. lbs.	.0763	.1166	.0738	.005	.0424	.0741
B.T.U./cu.ft. High.	----	----	341.	345.	1065.	----
B.T.U./cu.ft. Low.	----	----	342	293.6	962.1	----
% of C by Weight.	----	23.7	42.9	----	75.0	----
% of H by Weight.	----	----	----	100.	25.0	----

The cubic feet of gas generated is most accurately calculated from the chemical analysis of the coal, ash and gas. The total carbon appear-



Test Of A 400 H.P. Gas Producer.

ing in the gas is that fed to the generator in the form of coal; less that lost in the ash-pit. From the analysis of the ash the total refuse in the ash-pit may be calculated, for, knowing the per cent of ash in the coal by analysis, it is at once evident that the weight of refuse is

$$W_r = W(C_a + C_a C_r). \quad (1)$$

Where

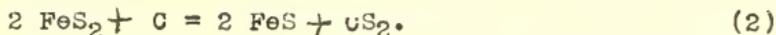
$W_r$  = the weight in pounds, of refuse in the ash-pit.

$W$  = the weight of coal used.

$C_a$  = the per cent of ash, plus half the per cent of sulphur in the coal.

$C_r$  = the per cent of carbon in the refuse.

Practically all of the sulphur in this coal occurs as iron pyrites ( $FeS_2$ ). Upon heating without excess air, the following changes take place,



The carbon disulphide passes off in the gas, while the iron sulphide remains in the ash to form a fusible clinker. From the equation it can be seen that half the sulphur in the coal stays in the ash while the other half passes off in the gas.

The carbon contained in the gas is found in



### Test Of A 400 H.P. Gas Producer.

the carbon dioxide, carbon monoxide and methane.

The weight of carbon in pounds, in one cubic foot of the gas =  $W_g = (.11665 \times .273 \times C_1) + (.07381 \times .429 \times C_2) + (.0424 \times .75 \times C_3)$ . (3)

Where

$W_g$  = the weight in pounds of carbon in one cubic foot of the gas.

$C_1$  = the proportion by volume of  $CO_2$  in the gas.

$C_2$  = the proportion by volume of  $CO$  in the gas.

$C_3$  = the proportion by volume of  $CH_4$  in the gas.

The constants refer to those in the table on

page 34. The carbon appearing in combination

with the sulphur as carbon disulphide is for the present, neglected. Simplifying Eq.(3),

$$W_g = .031845C_1 + .03166 C_2 + .031785 C_3. \quad (4)$$

The weight in pounds of carbon from the coal actually gasified is

$$W_a = W C_c - W C_a C_r = W_g. \quad (5)$$

Where

$W_a$  = weight of carbon actually gasified.

$W$  = weight of coal burned.

$C_c$  = the percent of carbon in the coal.

$C_a$  = the percent of ash plus half the percent of



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sulphur in the coal.

$\%_R$  = the percent of carbon in the refuse.

$W_S$  = the weight of carbon combined with the sulphur.

From equation (2) it may be seen that carbon combines with sulphur in the proportion of 12 to 64; that is, 3 pounds of carbon will combine with 16 pounds of sulphur. Now the weight of sulphur combining with the gas =  $W'_S = W C_S \div 2$ . (6)

Where

$W'_S$  = weight of sulphur in the carbon disulphide.

$W$  = the weight of coal used.

$\%_S$  = the percent of sulphur in the coal.

Then the weight of carbon combining with this sulphur =  $W_S = (W \%_S \div 2) \cdot 3/16 = 3 W \%_S \div 32$ . (7)

Substituting this value of  $W_S$  in Eq.(5),

$$W_a = W C_C - W C_a C_R - 3/32(W C_S). \quad (8)$$

Knowing the weight of carbon gasified and the weight of carbon in one <sup>u</sup>cubic foot of the gas, the cubic feet of gas generated =

$$V = W_a \div W_g. \quad (9)$$

It is believed that such losses as dust, absorption of gases by the scrubber water, etc., may be neg-



### Test Of A 400 H.P. Gas Producer.

lected as refinements not in keeping with the other observations.

In making the calculations of the air supplied it is assumed that there is no nitrogen in the coal and since this assumption is made, it is evident that all the nitrogen appearing in the gas must enter the system as a constituent of the air used. This air comes in thru the economizer, and with the steam. The amount of air entering thru the economizer is so large compared with that entering with the steam that it is assumed all the air enters the system thru the economizer. The leakage is negligible since the gas is under pressure. The weight of air can now be calculated from the equation,

$$A = .07411 V C_n + .768. \quad (10)$$

Where

A = the weight of dry air in pounds.

$C_n$  = the percent by volume of nitrogen in the gas.

V = the cubic feet of gas generated.

.07411 = the weight of a cubic foot of nitrogen.

.768 = the proportion by weight of nitrogen in the air. Simplifying Eq. (10),



Test Of A 400 H.P. Gas Producer.

$$A = .0965 V C_n. \quad (11)$$

The cubic feet of dry air is

$$A_v = 13.107 \times .0965 V C_n. \quad (12)$$

Where

13.107 is the volume in cubic feet, of one pound of air under standard conditions.

Simplifying the above expression,

$$A_v = 1.2643 V C_n. \quad (13)$$

In making the calculations for the amount of steam used, the assumption is made that the moisture is supplied to the generator in the following ways; 1st., by the steam, 2nd., as moisture in the coal, 3d., as aqueous vapor in the air. All the hydrogen in the gas is supplied to the system as moisture (less that in the volatile combustible in the coal). Now the amount of hydrogen supplied by the moisture is

$$H_m = H - H_v. \quad (14)$$

Where

$H_m$  = the hydrogen due to the moisture supplied.

$H$  = the total hydrogen in the gas.

$H_v$  = the hydrogen due to the volatile combustible in the coal.



### Test Of A 400 H.P. Gas Producer.

$$H_V = W (C_V + 4). \quad (15)$$

Where

$W$  = the weight of coal burned.

$C_V$  = the percent of volatile combustibile in the coal. ( 1/4 of the volatile combustibile by weight is hydrogen, assuming the volatile combustibile to be  $CH_4$  ).

The hydrogen in the gas due to the steam from the steam blower may now be calculated from the equation,

$$H_B = H - W(C_m + 9) - M_a. \quad (16)$$

Where

$W(C_m + 9)$  = the weight of hydrogen supplied by the moisture in the coal.

$C_m$  = the percent of moisture in the coal by weight.

$M_a$  = the weight of moisture in the air used.

$$M_a = H_a A_v K. \quad (17)$$

Where

$H_a$  = the relative humidity of the air at the temperature of the room.

$A_v$  = the cubic feet of air used.

$K$  = the weight in pounds, of moisture which will saturate one cubic foot of air at the observed



Test Of A 400 H.P. Gas Producer.

temperature of the room.

Substituting this value of  $M_a$  in Eq.(16),

$$H_g = H - W(C_m + 9) - H_a A_v K. \quad (18)$$

$$\text{The pounds of steam used} = W_g = 9 H_g. \quad (19)$$

since hydrogen is one-ninth by weight of water.

By means of the air tables and, knowing the relative humidity of the gas at its temperature in the factory mains, the weight of aqueous vapor taken up by the gas may be calculated. If

$H_g$  = the relative humidity of the gas,

$\lambda_g$  = the weight in pounds of moisture necessary to saturate one foot of the gas at the observed temperature of the gas,

$V$  = the cubic feet of gas,

$M_g$  = the total weight of aqueous vapor in the gas,

$$\text{Then } M_g = H_g V \lambda_g. \quad (20)$$

With the preceding derived formulas and the data taken during the test, Tables 3,4,5 and 6 have been calculated. To show the method of calculating these tables and the application of the preceding formulas, the numerical calculations for run No.7 is here given:-

Length of the test = 8 hours.



Test Of A 400 H.P. Gas Producer.

Total coal used = 2375 lbs.

Coal used per hour = 296.9 lbs.

Moisture in the coal, by weight = 2.2%.

Total moisture in the coal =  $2375 \times .022 = 52.2$  lbs.

Volatile combustible in the coal by weight = 4.4%.

Total volatile combustible in the coal =

$2375 \times .044 = 104.5$  lbs.

Ash in the coal by weight = 11.2%.

Total ash in the coal =  $2375 \times .112 = 266$  lbs.

Sulphur in the coal by weight = 1.9%.

Total sulphur in the coal =  $2375 \times .019 = 45.1$  lbs.

Total sulphur in the refuse =  $45.1 + 2 = 22.6$  lbs.

Total sulphur in the gas =  $45.1 + 2 = 22.6$  lbs.

Fixed carbon in the coal by weight = 80.3%

Total fixed carbon in the coal =

$2375 \times .803 = 1907.3$  lbs.

Total refuse in the ash-pit =  $2375 (.112 +$

$.112 \times .0872) = 316$  lbs. (from Eq.1).

Total carbon in the ash-pit =  $316 - (266 + 22.6)$

= 27 lbs.

Total carbon in the carbon disulphide =

$(3 \times 2375 \times .019) + 32 = 4.2$  lbs. (from Eq.7).

Total carbon disulphide =  $16/3 \times 4.2 = 26.8$  lbs.



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Total carbon in the volatile combustible =

$$104.5 \times .75 = 78.3 \text{ lbs.}$$

Total hydrogen in the coal = total hydrogen in

$$\text{the volatile combustible} = 104.5 - 78.3 = 26.1 \text{ lbs.}$$

$$\text{Total carbon in the coal} = 1907.3 + 78.3 = 1985 \text{ lbs.}$$

$$\text{Area of the grate} = 28.2744 \text{ sq.ft.}$$

Coal burned per sq.ft. of grate per hour =

$$296.9 \div 28.2744 = 10.5 \text{ lbs.}$$

Calculations of the gas, steam and air:-

$$\text{Total carbon in the gas} = 1985 - 4.2 - 27 = 1954 \text{ lbs.}$$

$$\text{Carbon in one cubic foot of the gas} = .031845 \times$$

$$.0658 + .031664 \times .2137 + .031785 \times .017 =$$

$$.009402 \text{ lbs. (from Eq.4)}$$

$$\text{cubic feet of gas generated} = 1954 \div .009402 =$$

$$207820 \text{ (from Eq.9)}$$

$$\text{Weight of nitrogen in the gas} = .07411 \times 207820$$

$$\times .5446 = 8345.86 \text{ lbs.}$$

$$\text{Weight of air used} = 8345.86 \div .768 = 10867 \text{ lbs.}$$

$$\text{Volume of air used} = 10867 \times 13.107 = 142424 \text{ cu.ft.}$$

$$\text{Air per pound of coal} = 142424 \div 2375 = 59.9 \text{ cu.ft.}$$

$$\text{Total hydrogen in the gas} = 207820 \times .1589 \times$$

$$.005 = 235.5 \text{ lbs.}$$

Deducting the hydrogen in the coal, hydrogen due



### Test Of A 400 H.P. Gas Producer.

to moisture =  $235.5 - 26.1 = 209.4$  lbs.

Total steam supplied =  $209.4 \times 9 = 1884.6$  lbs.

Moisture in the air =  $.45 \times .00221 \times 142424 =$   
78.2 lbs.

Steam supplied =  $1884.6 - (78.2 + 52.2) = 1754.2$  lbs.

Steam per hour =  $1754.2 \div 8 = 219.3$  lbs.

Gas per pound of coal =  $207820 \div 2375 = 87.5$  lbs.

Steam per pound of coal =  $1754.2 \div 2375 = .739$  lbs.

Moisture in the gas =  $.7 \times 207820 \times .00225 =$   
327.3 lbs.

The pressure and temperature differances given in Tables 5 & 6 are calculated from the observed readings of temperatures and pressures. The corrected amount of water used in the various parts of the plant were read off from the calibration curves of the water meters. These meters were calibrated under the same conditions as they were used, the amount of water being varied from a maximum flow to the slip of the meter. The water run thru the meters in calibrating them was caught in tanks and weighed, the corresponding meter readings being taken. Knowing the amount of water used in the plant, the amount of



## Test Of A 400 H.P. Gas Producer.

coal burned, and the gas generated, the water used per unit of coal or gas is easily calculated, and may be found in Table 6.

**Heat Distribution:-** This important part of a producer test is often the deciding point in the acceptance or not of a producer installation. However well a producer may make gas, its efficiency may not at all be sufficient to warrant installing, or even running. The conditions under which the gas is to be used must be such that the producer may be as economical if not more so, than any other method of fuel burning. The question of convenience and simplicity is also an item which bears consideration.

The producer gets its heat in two ways; first, from the fuel, either coal, water or air supplied to the producer; second, heat returned to the generator by the air and steam. The total heat supplied primarily, comes from the first source, but the efficiency of the producer largely depends upon the heat returned by the second source. In the following discussion and calculations, all amounts of heat are figured above 62 Deg.F., it



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being assumed that the coal fed to the generator is at this temperature. The lower, or effective heating value of the gas is used, because in burning the gas in the factory, the products of combustion is not cooled back to the original temperature of the gas. The higher heating value of the coal is used, since, in bringing about the change from solid to gaseous fuel, the sensible heat of the coal is rendered latent in the gas.

Efficiencies:- The actual efficiency of the producer may be stated as the ratio of the latent heat in the gas to the total heat supplied by the fuel. This is also known as the cold gas efficiency. The commercial efficiency is the ratio of the latent heat in the gas (plus the sensible heat above 62 Deg.F.) to the heat supplied to the producer in the form of fuel plus the heat equivalent of the energy received by the producer from an outside source. The grate efficiency is the ratio of the B.T.U. of the coal burned in the producer to the B.T.U. of the fuel supplied to the producer. In some cases, as in this case, there is no grate in the generator, but this efficiency



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will represent the ash-pit efficiency which is probably a more appropriate name for it. The efficiency of the economizer is the ratio of the heat actually taken up by the air, to the maximum amount which could be taken up by the air. The efficiency of an economizer depends altogether on its design. To obtain the highest efficiency from an economizer, all exposed surfaces should be well lagged, to reduce the radiation loss. The size must be such that there will be ample area for the heat transfer, the velocity of the gas and air thru the economizer must be low enough to allow the heat transfer to take place. As the heat transfer depends directly upon the temperature difference, for maximum efficiency the economizer should work on the counter current principle and should have as much surface exposed to the scrubbing action of the gases as possible, without creating an excessive resistance to the flow of the gases.

The heat supplied to the producer by the coal, is the weight of the coal in pounds times its B.T.U. per pound. The heat supplied by the steam is the



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weight of steam used times the total heat in the steam at the observed pressure and quality. A steam separator is used to take all the entrained moisture from the steam, and a throttling calorimeter placed on the steam line showed practically dry steam. The velocity of the steam thru the steam separator was very low, and this accounted for its high efficiency. The heat supplied by the air is the weight of the air times its specific heat times the difference in temperature (above 62 Deg. F.). The total heat supplied may be represented by the equation,

$$H = H_C + H_S + H_A. \quad (21)$$

Where

$H$  = the total heat supplied.

$H_C$  = the heat supplied by the coal.

$H_S$  = the heat supplied by the steam.

$H_A$  = the heat supplied by the air.

$$H_C = W h. \quad (22)$$

Where

$W$  = the weight of coal used in pounds.

$h$  = the heat value of one pound of coal.

$$H_S = W_S (x r + q) \quad (23)$$



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Where

$x$  = the quality of the steam.

$r$  = the latent heat of evaporation, in B.T.U. of the steam at the observed pressure.

$q$  = the heat of the liquid, in B.T.U. of the steam at the observed pressure.

$W_g$  = the weight of steam used in pounds.

$$H_a = W_a S_a (t_r - 62). \quad (24)$$

Where

$W_a$  = the weight of air in pounds.

$S_a$  = the specific heat of the air = .238.

$t_r$  = the temperature of the room, in Deg.F.

Substituting these values in Eq.(21),

$$H = Wh + W_g (xr + q) + W_a S_a (t_r - 62). \quad (25)$$

The latent heat in the gas is expressed by the equation,

$$H_g = V_g h_g. \quad (26)$$

Where

$H_g$  = the total heat in the gas.

$V_g$  = the volume in cubic feet of gas generated.

$h_g$  = the effective heat value of one cubic foot of the gas.

Then the actual efficiency of the producer is



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$$E_a = V_g h_g + (W_h + W_s(xr + q) + W_a S_a(t_r - 62)) \quad (27)$$

If  $S_h$  is the sensible heat in the gas above 62 Deg. F., and B the heat equivalent of energy supplied to the producer from an outside source, then the commercial efficiency of the producer may be expressed by the equation,

$$E_c = (V_g h_g + S_h) + (W_h + W_s(xr + q) + W_a S_a(t_r - 62) + B) \quad (28)$$

When taking the observed data the energy furnished from an outside source was taken as kilowatt hours.

$$\text{Then } B = 3412 \text{ K.} \quad (29)$$

Where

K = the number of kilowatt hours used.

3412 = the heat equivalent of one K.W.Hr.

Substituting this value of B in Eq. (28),

$$E_c = (V_g h_g + S_h) + (W_h + W_s(xr + q) + W_a S_a(t_r - 62) + 3412 \text{ K}). \quad (30)$$

The efficiency of the grate may be expressed by the equation,

$$E_g = (W_h - 14500 W_g) \div W_h \quad (31)$$

Where

$W_h$  = the heat supplied by the coal.

$W_g$  = the weight in pounds of carbon in the ash-pit.

14500 = the heat of combustion of one pound of



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carbon.

The efficiency of the economizer may be expressed by the equation,

$$E_e = \frac{W_a S_a (t - t_1) + W_g S_g (t_0 - t_1)}{(t - t_1) + (t_0 - t_1)} \quad (32)$$

Where

$t$  = temperature of the air leaving economizer.

$t_1$  = temperature of the air entering economizer.

$t_0$  = temperature of the gas entering economizer.

Equation (32) is true since there is always more available heat in the gas than can be taken up by the air.

The specific heat of a mixture of gases, varies as the proportion of the gases in the mixture.

The specific heat of a mixture of gases is obtained by multiplying the specific heat of each separate gas by its proportional weight in the mixture and adding the results. Gas generated from a gas producer generally contains carbon dioxide, carbon mono-ide, hydrogen, methane and nitrogen.

The specific heats due to,

$$CO_2 = .2164 \times .1164 C_1 = .0252 C_1.$$

$$CO = .2479 \times .0738 C_2 = .0183 C_2.$$



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$$H = 3.4046 \times .0053 C_3 = .1702 C_3.$$

$$CH_4 = .593 \times .0424 C_4 = .0251 C_4.$$

$$N = .244 \times .0741 C_5 = .0181 C_5.$$

Where  $C_1, C_2, etc.$  refer to the percents by volume of the constituents in the gas. The specific heats in the previous equations refer to those at constant pressure, per pound of the gas. These specific heats have been determined by experiment and there seems to be such a difference with the different experimenters, that it is not thought advisable to use them in these calculations.

Table 7 gives the results of the calculations of the heat distribution. As an example of the method of the calculations of this table, those of run No.7 is here given:-

$$\text{Heat supplied by the coal} = 296.9 \times 12934 = 3,840,105 \text{ B.T.U.}$$

$$\text{Heat supplied by the steam} = 219.3 (922 + 244 - 30) = 250,221 \text{ B.T.U.}$$

$$\text{Heat supplied by the air} = 10,867 \times .238 (72 - 62) = 3,233 \text{ B.T.U.}$$

$$\text{Heat supplied as fuel} = 3,840,105 + 250,221 + 3,233 = 4,093,559 \text{ B.T.U.}$$

$$\text{Heat supplied as energy} = 136.25 \times 3412 =$$



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464,885 B.T.U.

Total heat supplied, fuel plus energy =  $4,093,559 + 464,885 = 4,558,444$  B.T.U.

Percent of the heat in the fuel supplied by the coal =  $3,840,105 \div 4,093,559 = 93.9 \%$ .

Percent of heat in the fuel supplied by the steam =  $250,221 \div 4,093,559 = 6.05 \%$ .

Percent of heat in the fuel supplied by the air =  $3,233 \div 4,093,559 = 0.05 \%$ .

Percent of the total heat supplied as energy =  $464,885 \div 4,558,444 = 10.2 \%$ .

Latent heat in the gas, high value, from analysis =  $25,978 \times 145.8 = 3,782,592$  B.T.U.

Latent heat in the gas, low value, from analysis, =  $25,978 \times 135.9 = 3,530,410$  B.T.U.

Latent heat in the gas, high value, from calorimeter =  $25,978 \times 144.8 = 3,761,614$  B.T.U.

Latent heat in the gas, low value, from calorimeter =  $25,978 \times 134.9 = 3,507,030$  B.T.U.

Heat lost as carbon in the ash-pit =  $14,500 \times 27 = 49,000$  B.T.U.

Sensible heat in the gas at the temperature of the furnace =  $4,093,559 - 3,530,410 = 563,149$  B.T.U.

Heat taken up by the air in the economizer =



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$$(10,867 \div 8) (238 \times 325) = 105,040 \text{ B.T.U.}$$

Percent of heat as fuel returned by the economizer  
 $= 105,040 \div 4,093,559 = 2.62 \%$

Percent of the sensible heat returned by the  
 economizer  $= 105,040 \div 563,149 = 18.6 \%$

Heat taken up by the water in the scrubber =  
 $61,158 \div 8) (80 - 38) = 321,029 \text{ B.T.U.}$

Heat added by the exhauster compressing the gas  
 $= 25,978 \times .0198(92 - 50) = 20,393 \text{ B.T.U.}$

Ratio of compression heat to the energy heat =  
 $20,393 \div 464,885 = 4.39$

Heat lost to the water cooled top =  $(8000 \div 8,$   
 $(138 - 38) = 100,000 \text{ B.T.U.}$

Radiation plus unaccounted for losses =  $563,149 -$   
 $(105,040 - 100,000 - 321,029) = 37,080 \text{ B.T.U.}$

Actual efficiency,  $E_a = 3,530,410 \div 4,093,559 = 86.3\%$

Commercial efficiency,  $E_c = 3,530,410 \div 4,556,444 =$   
 $77.5 \%$

Grate efficiency,  $E_g = (3,840,105 - 49,000) \div$   
 $3,840,105 = 98.73 \%$

Economizer efficiency,  $E_e = (397 - 72) \div (1262 - 72)$   
 $27.8 \%$

Ratio of  $E_a$  to  $E_c = 77.5 \div 86.3 = 89.9.$



TABLE 3.  
COAL AND ASH DATA.

NUMBER OF TEST	1	2	3	4	5	6	7	8	AVERAGE.
TOTAL COAL USED, LBS.	337.5	125.0	265.0	1000	312.5	237.5	237.5	212.5	2037.6
COAL PER HOUR, LBS.	375.0	156.3	331.2	125.0	390.6	296.9	296.9	265.6	264.7
MOISTURE IN THE COAL, PERCENT BY WT.	2.5	2.5	2.5	2.5	2.5	2.2	2.2	2.2	2.4
VOLAT. COMB. IN THE COAL, PERCENT BY WT.	5.5	5.5	5.5	5.5	5.5	4.4	4.4	4.4	5.1
ASH IN THE COAL, PERCENT BY WT.	14.7	14.7	14.7	14.7	14.7	11.2	11.2	11.2	13.4
SULPHUR IN THE COAL, PERCENT BY WT.	1.2	1.2	1.2	1.2	1.2	1.9	1.9	1.9	1.46
FIXED CARBON IN THE COAL, PERCENT BY WT.	76.1	76.1	76.1	76.1	76.1	80.3	80.3	80.3	77.7
B.T.U. PER LB. OF COAL AS FIRED.	12613	12613	12613	12613	12613	12934	12934	12934	12733
CARBON IN ASH-PIT, PERCENT BY WT. OF REFUSE	5.60	4.83	6.47	6.80	5.90	11.20	8.72	9.10	7.09
TOTAL MOISTURE IN THE COAL, LBS.	84.4	31.2	66.2	25.0	78.1	52.2	52.2	46.7	54.5
TOTAL VOLAT. COMB. IN THE COAL, LBS.	185.6	68.8	146.0	55.0	172.0	104.5	104.5	93.5	116.2
TOTAL ASH IN THE COAL, LBS.	496.1	183.7	382.5	147.0	459.4	266.0	266.0	238.0	305.7
TOTAL SULPHUR IN THE COAL, LBS.	40.5	15.0	31.8	12.0	37.5	45.1	45.1	40.4	33.4
TOTAL SULPHUR IN THE REFUSE, LBS.	20.3	7.5	15.9	6.0	18.7	22.5	22.6	20.2	16.7
TOTAL FIXED CARBON IN THE COAL, LBS.	2568.4	951.3	2016.7	761.0	2378.1	1907.3	1907.3	1706.4	1774.6
TOTAL REFUSE IN THE ASH-PIT, LBS.	547.0	201.0	429.0	164.0	503.0	325.0	316.0	284.0	346.1
TOTAL CARBON IN THE ASH-PIT, LBS.	31.0	10.0	24.0	11.0	25.0	36.0	27.0	26.0	23.9
TOTAL CARBON IN THE $CS_2$ , LBS.	3.8	1.4	3.0	1.2	3.5	4.2	4.2	3.8	3.14
TOTAL $CS_2$ (CARBON DISULPHIDE), LBS.	24.1	8.9	18.9	7.1	22.3	26.8	26.8	24.0	19.9
TOTAL CARBON IN THE VOLAT. COMB., LBS.	139.2	51.6	109.5	42.0	129.0	78.3	78.3	70.2	87.3
TOTAL HYDROGEN IN THE COAL, LBS.	46.4	17.2	36.5	14.0	43.0	26.1	26.1	23.4	29.1
TOTAL CARBON IN THE COAL, LBS.	2707.0	1003.0	2126.0	803.0	2507.0	1995.0	1995.0	1776.0	1861.5
COAL PER SQ. FT. OF GRATE PER HR., LBS.	13.3	5.5	11.7	4.4	13.8	10.5	10.5	9.3	9.9
STEAM PER LB. OF COAL, LBS.	.564	.518	.649	.555	.682	.786	.739	.697	.648
AIR PER POUND OF COAL, CU. FT.	6.1	5.94	5.94	6.32	6.91	6.17	5.9.9	5.7.6	6.0.4
GAS PER POUND OF COAL, CU. FT.	83.6	93.6	84.5	85.9	94.0	89.3	87.5	84.5	86.6

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TABLE 4  
GAS, AIR AND STEAM DATA.

	1	2	3	4	5	6	7	8	AVERAGE
NUMBER OF RUN									
CO <sub>2</sub> IN THE GAS, PERCENT BY VOL.	7.00	5.93	6.14	7.92	5.43	6.57	6.58	7.8	6.67
CO IN THE GAS, PERCENT BY VOL.	21.09	22.19	21.60	19.34	20.23	20.62	21.37	20.80	20.90
H IN THE GAS, PERCENT BY VOL.	13.24	13.92	15.01	12.81	15.33	16.57	15.89	15.15	14.74
CH <sub>4</sub> IN THE GAS, PERCENT BY VOL.	1.79	1.79	1.82	1.75	0.93	1.70	1.70	1.79	1.66
N IN THE GAS, PERCENT BY VOL.	57.84	56.16	55.43	58.18	58.08	54.54	54.46	54.46	56.16
HUMIDITY OF THE GAS, PERCENT	55	73	72	76	80	68	70	73	74.6
WT. OF CARBON IN 1 CU.FT. OF GAS, LBS.	0.09473	0.09484	0.09343	0.09202	0.09431	0.09162	0.09402	0.09639	0.09170
WT. OF HYDROGEN IN 1 CU.FT. OF GAS, LBS.	0.00984	0.01025	0.01093	0.00954	0.01018	0.01174	0.01123	0.01099	0.01066
WT. OF NITROGEN IN 1 CU.FT. OF GAS, LBS.	0.42865	0.41620	0.41079	0.43117	0.43043	0.40420	0.40360	0.40360	0.41580
CALCULATED B.T.U. PER CU.FT., HIGH.	136.6	143.8	140.8	128.8	131.8	148.6	145.8	142.3	139.9
CALCULATED B.T.U. PER CU.FT., LOW.	129.0	135.8	134.5	120.4	123.0	135.4	135.9	132.7	130.8
FROM CALORIMETER, B.T.U. PER CU.FT., HIGH.	147.4	142.4	146.5	142.9	140.0	153.8	144.8	140.0	144.8
FROM CALORIMETER, B.T.U. PER CU.FT., LOW.	139.8	134.4	136.2	134.5	131.2	143.6	134.9	130.4	136.7
CUBIC FEET OF GAS, TOTAL.	281075	104880	224660	85938	293900	212280	207820	180100	197842
CUBIC FEET OF GAS PER HOUR.	31342	13069	28082	10742	36738	26535	25978	21872	24730
TOTAL MOISTURE IN THE GAS, LBS.	719.3	229.0	364.0	196.0	529.0	424.8	327.3	295.4	332.4
TOTAL CARBON IN THE GAS, LBS.	2672.0	991.6	2098.0	790.8	2478.5	1945.0	1854.0	1736.0	1834.0
TOTAL AIR USED, LBS.	15743	5666	12016	4825	16472	11172	10867	9418	10711
TOTAL AIR USED, CU. FT.	206330	74260	157483	63237	215884	146420	142424	122423	140380
TOTAL MOISTURE IN THE AIR, LBS.	94.5	41.0	96.0	31.7	96.0	87.7	78.2	64.1	74.4
TOTAL HYDROGEN IN THE GAS, LBS.	272.56	107.16	245.6	82.0	299.2	249.2	235.6	197.9	209.7
HYDROGEN IN GAS DUE TO MOISTURE, LBS.	231.2	90.0	209.1	68.0	256.2	223.1	209.4	174.5	180.6
TOTAL MOISTURE SUPPLIED, LBS.	2080.8	720.0	1881.9	612.0	2305.8	2007.9	1884.6	1570.5	1633.0
MOISTURE SUPPLIED BY AIR & COAL, LBS.	178.9	72.2	162.0	56.7	173.1	139.9	130.4	110.8	128.9
TOTAL STEAM SUPPLIED LBS.	1901.9	647.8	1719.9	555.3	2132.7	1868.0	1754.2	1459.7	1496.5
STEAM PER HOUR, LBS.	211.3	81.0	215.0	69.4	266.6	233.5	219.3	182.5	187.1

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TABLE 5.  
TEMPERATURE DATA, IN DE6. FAH.

	1	2	3	4	5	6	7	8	AVERAGE
NUMBER OF RUN									
OUTSIDE AIR.	36.	49.	50	52	55.	51	49	54	50
AIR IN ROOM.	62	66	73	68	60	70	72	72	68
RISE IN TEMP. OF AIR IN ROOM	26	17	23	16	5	19	23	18	18
AIR TO ECONOMIZER.	62	66	73	68	60	70	72	72	70
AIR FROM ECONOMIZER	378	323	366	315	356	404	397	362	363
RISE IN TEMP. OF AIR IN ECONOMIZER	316	257	293	247	296	334	325	290	295
FURNACE	1300	1093	1198	1098	1244	1312	1462	1316	1221
GAS TO ECONOMIZER	1300	1093	1198	1090	1244	1312	1462	1316	1221
GAS FROM ECONOMIZER	616	426	569	375	594	568	623	596	546
DROP IN TEMP. OF GAS IN ECONOMIZER	684	667	629	665	650	744	639	720	675
GAS TO SCRUBBER	163	305	161	260	143	157	170	156	189
DROP IN TEMP. OF GAS TO SCRUBBER	521	362	468	405	507	587	469	564	485
GAS FROM SCRUBBER	51	50	49	51	54	60	50	50	52
DROP IN TEMP. OF GAS IN SCRUBBER.	112	255	112	209	89	107	120	106	139
GAS IN FACTORY MAINS.	107	102	92	106	92	91	92	90	97
RISE IN TEMP. OF GAS IN EXHAUSTER	56	52	43	55	38	41	42	40	46
WATER TO SCRUBBER	35	35	34	34	36	37	38	38	36
WATER FROM SCRUBBER.	75	57	62	49	85	89	80	78	72
RISE IN TEMP. OF WATER IN SCRUBBER	40	22	28	15	49	52	42	40	36
WATER TO WATER COOLED TOP.	35	35	34	34	36	37	38	38	36
WATER FROM WATER COOLED TOP.	109	95	157	68	125	101	138	104	112
RISE IN TEMP. OF WATER IN TOP.	74	60	123	39	89	64	100	69	76
WATER TO ASH-PIT SEAL.	109	95	157	68	125	101	138	104	112.
WATER FROM ASH-PIT SEAL.	92.	82	117	78	112	89	104	91	96
DROP IN TEMP. OF WATER IN ASH-PIT SEAL	17	13	40	-10	13	12	34	13	17

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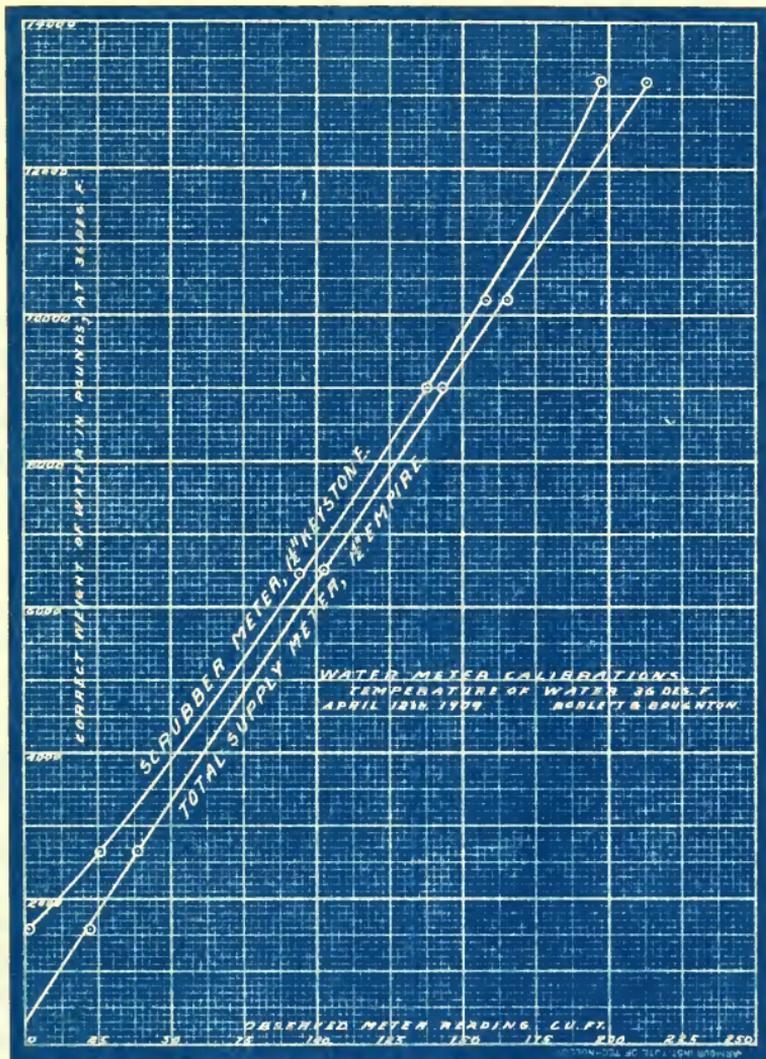
TABLE 6

## PRESSURE, POWER AND WATER DATA.

	1	2	3	4	5	6	7	8	AVERAGE
NUMBER OF RUN.	2.04	2.18	1.50	2.36	2.50	2.03	1.60	2.02	2.03
PRESSURE IN BOTTOM OF GENERATOR, INS. H <sub>2</sub> O.	0.36	0.16	0.46	0.17	0.35	0.22	0.10	0.14	0.25
PRESSURE IN TOP OF GENERATOR, INS. H <sub>2</sub> O.	4.13	1.08	2.43	0.51	3.33	3.28	3.12	3.82	2.71
SUCTION OF EXHAUSTER, INS. H <sub>2</sub> O.	1.69	2.02	1.04	2.19	2.15	1.81	1.50	1.88	1.78
PRESSURE TO OVERCOME RESISTANCE OF FUEL BED, INS. H <sub>2</sub> O.	2.39	2.39	1.96	2.53	2.85	2.25	1.70	2.16	2.28
PRESSURE FURNISHED BY STEAM BLOWER, INS. H <sub>2</sub> O.	59.4	10.8	35.6	20.1	46.0	48.0	45.0	44.0	39.2
STEAM PRESSURE, LBS. PER SQ. IN.	2.82	2.75	2.75	2.63	2.50	2.63	2.50	2.54	2.64
PRESSURE IN FACTORY MAINS, LBS. PER SQ. IN.	22.83	22.32	22.21	22.92	22.86	22.36	22.52	22.23	22.36
BAROMETRIC PRESSURE, INS. Hg.	14.65	14.40	14.34	14.49	14.17	14.61	14.49	14.35	14.42
BAROMETRIC PRESSURE, LBS. PER SQ. IN.	105.750	64.000	1343.95	1222.37	687.47	562.97	691.59	812.56	1090.41
TOTAL WATER USED IN PLANT, LBS.	12,679	7,674	16,113	14,659	8,283	6,783	8,332	9,799	13,074
TOTAL WATER USED IN PLANT, GALS.	904.50	515.00	12,638.4	10,863.7	6,010.7	5,060.6	6,115.8	6,453.6	9,600.0
WATER USED IN SCRUBBER, LBS.	10830	6140	15195	13020	7472	6097	7369	7775	11511
WATER USED IN SCRUBBER, GALS.	15300	12800	8000	13600	8640	5691	8000	16720	13053
WATER USED IN WATER COOLED TOP, LBS.	1835	1534	959	1630	1041	685	964	2014	1565
WATER USED IN WATER COOLED TOP, GALS.	15300	12800	8000	13600	8640	5691	8000	16720	13053
WATER USED IN ASH-PIT SEAL, LBS.	1835	1534	959	1630	1041	685	964	2014	1565
WATER USED IN ASH-PIT SEAL, GALS.	3.8	6.1	6.1	14.7	4.5	6.6	4.1	6.5	6.55
TOTAL WATER PER POUND OF COAL, GALS.	3.2	4.9	5.7	13.0	3.9	5.9	3.7	5.6	5.74
WATER IN SCRUBBER PER POUND OF COAL GALS.	38.4	58.8	66.7	162.1	41.7	66.1	42.2	66.6	66.6
TOTAL WATER PER POUND OF COAL, LBS.	31.7	50.9	50.9	122.6	37.5	55.0	34.2	54.2	54.6
TOTAL WATER PER 1000 CU. FT. GAS., GALS.	45.6	73.2	71.4	170.5	48.2	73.9	46.7	76.1	79.5
TOTAL POWER USED, K. W. HRS.	1188.0	580.0	860.0	750.0	1100.0	990.0	1090.0	980.0	942.3
POWER USED PER HOUR, K. W. HRS.	132.00	72.50	107.50	93.75	137.50	123.75	136.25	122.50	116.5
POWER USED PER POUND OF COAL, K. W. HRS.	3.52	4.63	3.25	7.50	3.52	4.17	4.69	4.81	4.50
POWER USED PER 1000 CU. FT. OF GAS, K. W. HRS.	4.224	5.556	3.803	8.775	3.766	4.670	5.233	5.628	5.220
POWER USED PER TON OF COAL, K. W. HRS.	70.4	92.6	65.0	150.0	70.4	83.4	91.8	96.2	90.2

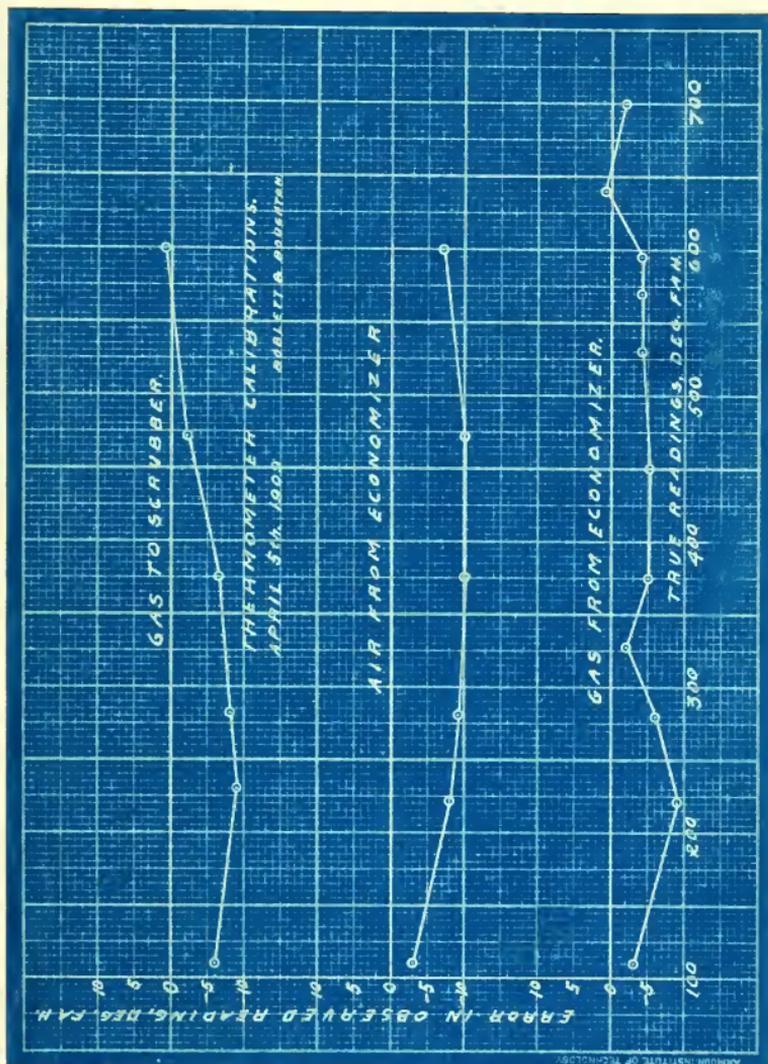
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## Test Of A 400 H.P. Gas Producer.



ANNALS  
OF THE ENTOMOLOGICAL SOCIETY OF AMERICA  
[PUBLISHED BIODIDACTICALLY]

## Test Of A 400 H.P. Gas Producer.



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INSTITUTE OF TECHNOLOGY  
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INSTITUTE OF TECHNOLOGY  
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TABLE 7

## HEAT DISTRIBUTION, IN B.T.U. PER HOUR, ABOVE 62° F.

NUMBER OF RUN.	1	2	3	4	5	6	7	8	AVERAGE
HEAT SUPPLIED BY THE COAL.	4,729,975	4,971,412	4,777,926	4,576,625	4,926,638	3,940,105	3,849,105	3,435,270	3,562,142
HEAT SUPPLIED BY THE STEAM.	242,043	91,157	244,606	79,491	304,324	266,657	250,221	209,470	210,675
HEAT SUPPLIED BY THE AIR.	0.0	674	3929	961	-490	2,659	3,233	2,802	1,709
HEAT SUPPLIED AS FUEL.	4,971,998	2,063,243	4,925,961	4,655,977	5,230,422	4,099,421	4,093,559	3,646,542	3,774,566
HEAT SUPPLIED AS ENERGY.	450,394	247,370	366,790	319,875	469,150	422,235	464,885	417,970	394,824
TOTAL HEAT SUPPLIED, FUEL PLUS ENERGY.	5,422,392	2,310,613	4,992,751	4,975,852	5,699,572	4,531,656	4,558,444	4,064,512	4,169,390
HEAT SUPPLIED AS COAL, PERCENT OF HEAT IN FUEL.	95.2	95.4	94.2	95.2	94.2	93.5	93.9	94.1	94.6
HEAT SUPPLIED AS STEAM, PERCENT OF HEAT IN FUEL.	4.8	4.4	5.5	4.7	6.0	6.45	6.05	5.73	5.45
HEAT SUPPLIED AS AIR, PERCENT OF HEAT IN FUEL.	0.0	0.2	0.3	0.3	-0.2	0.05	0.05	0.17	0.04
HEAT SUPPLIED AS ENERGY, PERCENT OF TOTAL HEAT.	8.3	10.7	8.2	16.4	8.2	9.3	10.2	10.3	10.2
LATENT HEAT IN THE GAS, HIGH VALUE, ANALYSIS.	4,291,307	1,879,322	4,662,749	1,383,570	4,842,069	3,863,496	3,782,572	3,203,458	3,462,200
LATENT HEAT IN THE GAS, LOW VALUE, ANALYSIS.	4,043,118	1,774,770	3,777,029	1,293,337	4,519,774	3,592,831	3,530,410	2,997,392	3,189,702
LATENT HEAT IN THE GAS, HIGH VALUE, CALORIMETER.	3,619,911	1,861,026	3,949,013	1,534,106	5,143,320	4,081,093	3,761,614	3,151,680	3,580,984
LATENT HEAT IN THE GAS, LOW VALUE, CALORIMETER.	4,201,612	1,756,474	3,247,649	1,444,799	4,920,026	3,810,426	3,507,030	2,935,565	3,355,061
HEAT LOST AS CARBON IN THE ASH-PIT.	49,933	18125	43,500	20000	45315	65250	49,000	47125	42140
SENSIBLE HEAT IN THE GAS, ABOVE FURNACE TEMP.	928,780	2,894,733	6,899,328	362,690	711,698	516,582	583,149	659,200	694,864
HEAT TAKEN UP BY AIR IN ECONOMIZER.	133,467	43,320	147,741	35,448	145,123	111,058	108,940	81,237	94,930
PERCENT OF HEAT, AS FUEL, RETURNED BY ECONOMIZER.	2.69	2.10	3.36	2.14	2.78	2.70	2.62	2.23	2.45
PERCENT OF SENSIBLE HEAT RETURNED BY ECONOMIZER.	14.4	16.0	16.2	9.9	20.4	21.6	18.6	12.3	16.3
HEAT TAKEN UP BY WATER IN SCRUBBER.	489,320	141,614	342,344	203,685	369,155	329,000	321,029	346,795	317,618
HEAT ADDED BY EXHAUSTER COMPRESSING GAS.	33,223	12,677	22,466	11,064	25,974	20,353	20,393	16,884	21,268
RATIO OF COMPRESSION HEAT TO ENERGY HEAT.	7.40	5.12	6.13	3.46	5.52	4.82	4.39	4.03	5.38
HEAT LOST TO WATER-COOLED TOP.	14,525	96,000	123,000	57,800	96,130	45,527	100,000	133,760	124,002
RADIATION PLUS UNACCOUNTED FOR LOSSES.	79,143	25,774	118,847	65,477	2,290	31,000	37,080	26,072	48,314
ACTUAL EFFICIENCY, $E_a$ .	81.4	85.9	85.3	78.2	86.3	87.3	86.3	84.3	85.6
COMMERCIAL EFFICIENCY, $E_c$ .	74.6	76.6	78.7	66.5	79.3	79.2	77.5	73.7	76.6
GRATE EFFICIENCY, $E_g$ .	98.95	99.08	98.96	98.73	99.08	98.30	98.73	98.63	98.79
ECONOMIZER EFFICIENCY $E_e$ .	29.0	29.5	30.5	30.3	28.6	30.9	31.5	27.8	29.8
RATIO OF $E_a$ TO $E_c$ .	91.7	89.3	92.3	83.8	91.9	90.8	89.9	87.5	89.5

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BOSTON

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AVERAGES	3375	9.7
TOTALS	3375	9.7

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MARCH 13 1909

LOG OF TEST RUN NO.2

ON GENERATOR NO. 2

TIME	FUEL USED LBS	TEMPERATURES DEG. FAHR										PRESSURES						METER READING			GAS ANALYSIS				HEATING VALUE OF THE GAS.						
		AIR					GAS					WATER						TOTAL WATER CU. FT.	SCRUBBER WATER CU. FT.	WATT METER 10 KW. HR.	PER CENT BY VOL.				FROM ANALYSIS BTU. PER CU. FT. HIGH	FROM ANALYSIS BTU. PER CU. FT. LOW					
		OUTSIDE	ROOM	FROM ECONOMIZER OF GENERATOR OR TO ECONOMIZER	FROM ECONOMIZER TO SCRUBBER	FROM SCRUBBER	IN FACTORY MAIN	WATER SUPPLY	FROM SCRUBBER	FROM WATER COOLED TOP TO ASH-PIT SEAL	FROM ASH-PIT SEAL	STEAM LBS./SQ. IN.	BOTTOM OF GENERATOR IN. OF WATER	TOP OF GENERATOR IN. OF WATER	SUCTION OF EXHAUSTER LESS THAT IN THE GEN- ERATOR IN. OF WATER	IN FACTORY MAINS LBS. & OZ.	BAROMETER INS. OF HG.				WATER	POWER	CO <sub>2</sub>	CO			H <sub>2</sub>	CH <sub>4</sub>	N <sub>2</sub>		
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31	
7:30		45	58	312	990	368	228		83	35	55	93	80	10		0.5	1.8	2-13		240379	1437	10302									
7:50		49	60	312	1080	370	252		90	35	54	122	89	12		0.4	1.3	2-10													
8:10		39	58	300	960	367	251		84	35	56	114	94	9		0.4	1.1	2-11	2938												
8:30		41	58	298	1060	374	266		97	35	56	129	92	8		0.3	1.2	2-12		240485	1536	10309	444	2404	18.6	177	5115	1642		146.3	
8:50		49	64	300	1130	398	286		98	35	56	86	91	4		0.3	1.2	2-10													
9:10		49	64	298	980	417	312		98	35	59	65	82	15		0.7	1.0	2-8													
9:30		50	64	320	1110	418	312		100	35	59	88	83	15		0.3	0.9	2-9		240600	1621	10316									
9:50		50	66	302	1160	408	306		100	35	58	133	82	10		0.3	0.5	2-12													
10:10		51	68	316	1245	466	357		100	35	58	125	91	16		0.3	1.0	2-10	2936	240700	1704	10322									
10:30		51	68	318	1130	456	341		101	35	62	105	92	13		0.0	1.1	2-9						6.4	23.0	14.1	212	5436	1497		113.7
10:50		51	68	322	1205	459	330		100	35	61	116	92	14		0.0	1.2	2-11													
11:10		49	67	316	1080	414	318		101	35	60	90	83	8		0.0	1.0	2-11													
11:30		48	67	316	1160	404	326		102	35	60	76	81	9		0.0	1.1	2-11		240827	1798	10330	64	21.3	13.4	1.44	57.46	1345			
11:50		50	68	320	1210	458	344		101	35	62	80	81	15		0.2	1.1	2-11													
12:10		52	69	320	1106	402	282		102	35	60	121	78	8		0.4	0.5	2-11													
12:30		53	69	318	1125	344	245		104	35	52	124	94	8		0.0	0.6	2-11		241070	1912	10338									
12:50		50	68	312	1020	410	310		109	35	52	67	75	10		0.0	1.1	2-7													
1:10		52	69	310	1100	444	332		107	35	59	60	73	15		0.0	0.8	2-7	2928												
1:30		48	69	314	1200	484	334		108	35	60	100	75	17		0.0	1.2	2-11		241105	1954	10345	54	20.6	7.00	2.20	64.38	1294		162.2	
1:50		51	68	332	1060	448	332		107	35	61	107	81	9		0.0	1.1	2-11													
2:10		53	68	322	1120	434	304		107	35	57	121	84	11		0.0	0.5	2-11													
2:30		50	68	330	1180	422	292		108	35	55	87	91	9		0.0	0.4	2-10		241330	2071	10357									
2:50		45	66	328	1080	480	270		110	35	54	55	70	7		0.0	0.6	2-11													
3:10		48	68	312	1040	349	244		110	35	52	58	62	9		0.0	0.5	2-11	2928												
3:30	1250	52	68	304	1090	342	228		112	35	50	74	65	9		0.0	0.4	2-10		241000	2193	10360	701	22.0	16.0	1.41	53.58	141.3		147.4	
AVERAGE & TOTALS		1250	494	660	314	1003	4214	297	102	35	571	948	824	10.8		1.6	0.92	2-104	2932	1021	756	580	5.9	22.2	13.9	1.79	56.21	143.8		142.4	

READINGS NOT CORRECTED

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INSTITUTE OF TECHNOLOGY  
BANGALORE

MARCH 19 1909

LOG OF TEST. RUN NO.3

ON GENERATOR NO.2

TIME	FUEL USED LBS	TEMPERATURES DEG. FAHR												PRESSURES						METER READING		GAS ANALYSIS					HEATING VALUE					
		AIR				GAS				WATER				PER CENT BY VOL.						OF THE GAS												
		OUTSIDE	ROOM	FROM ECONOMIZER OF GENERATOR OR TO ECONOMIZER	FROM ECONOMIZER TO SCRUBBER	FROM SCRUBBER TO FACTORY MAIN	WATER SUPPLY	FROM SCRUBBER	FROM WATER COOLED TOP TO ASHPIT SEAL	FROM ASHPIT SEAL	STEAM LBS./SQ. IN.	BOTTOM OF GENERATOR	IN OF WATER	TOP OF GENERATOR	SUCTION OF EXHAUSTER	LESS THAT IN THE GEN. ERATOR	IN OF WATER	IN FACTORY MAINS LBS. & OZ.	BAROMETER	IN. HG.	TOTAL WATER CU. FT.	SCRUBBER WATER CU. FT.	WATT METER 10 KW. HR.	CO <sub>2</sub>	CO	H <sub>2</sub>	CH <sub>4</sub>	N <sub>2</sub>	FROM ANALYSIS BTU PER CU. FT. HIGH	FROM ANALYSIS BTU PER CU. FT. LOW	FROM CALORIMETER BTU PER CU. FT. HIGH	
7:40	50	69	345	243	573	148	49	90	35	74	154	110	36	10	0.5	2.4	2-11	241473	2251	10926												
8:00	51	68	340	260	554	148	49	90	34	60	157	115	39	11	00	2.5	2-9															
8:20	51	68	340	230	566	132	48	88	35	58	162	104	39	2.0	02	2.5	2-10															
8:40	48	68	338	241	563	130	48	89	34	58	161	118	43	3.0	1.0	2.5	2-10	241928	2553	10936	6.6	21.4	1536	21.6	5493	1489	1388					
9:00	49	67	332	232	540	140	48	90	34	59	162	112	42	2.9	1.6	1.7	2-10															
9:20	48	69	332	244	546	138	47	91	34	58	162	114	41	1.3	01	2.6	2-10															
9:40	48	72	352	273	556	132	48	91	34	58	173	122	40	1.4	00	2.8	2-10	241211	2853	10947												
10:00	49	71	344	225	538	140	48	92	34	60	172	114	40	2.1	1.0	2.1	2-13															
10:20	53	70	354	214	566	134	48	91	34	60	178	114	4.2	1.70	02	3.1	2-8															
10:40	47	72	356	213	602	133	48	88	34	59	183	125	4.2	1.3	0.5	2.8	2-10	2920	242400	3156	10957	6.2	19.1	1394	29.9	5777	1451	1348	1502			
11:00	52	76	354	250	558	135	48	90	34	61	176	124	4.0	1.0	00	1.5	2-11															
11:20	51	73	362	288	601	144	48	90	34	61	180	125	4.4	1.5	00	3.4	2-10															
11:40	50	75	368	260	611	138	48	90	34	61	180	125	3.4	1.0	01	3.3	2-13	242656	3377	10968												
12:00	51	75	368	243	552	220	49	95	34	64	182	124	20	1.5	1.0	00	2-15															
12:20	51	72	378	260	466	270	50	96	34	56	180	127	15	2.0	1.5	00	2-12															
12:40	50	70	344	248	539	136	49	97	34	57	174	125	3.8	0.6	01	2.4	2-11	2920	242940	3654	10978	5.7	20.5	1476	0.74	58.30	129.7	120.4	145.6			
1:00	60	72	347	260	572	136	49	94	34	61	183	126	4.0	1.1	01	2.7	2-10															
1:20	49	74	362	240	618	130	49	91	34	63	182	127	5.0	2.7	00	3.7	2-9															
1:40	52	77	388	246	640	146	49	94	34	66	175	126	4.0	0.0	04	2.2	2-4	2921	243151	3805	10989	6.7	23.3	1493	2.00	5227	1668	146.3	139.8			
2:00	51	77	380	230	590	146	49	94	34	68	111	119	3.6	1.4	0.4	1.0	2-11															
2:20	50	72	382	240	585	146	49	95	34	68	114	112	3.6	1.5	0.5	2.2	2-14															
2:40	51	72	360	220	542	181	49	94	34	62	96	110	3.5	1.6	00	0.0	2-13	243395	4121	11000												
3:00	50	75	364	230	560	180	49	95	34	64	120	112	4.0	1.8	0.4	1.0	2-12															
3:20	53	73	358	252	538	249	49	94	34	63	95	105	3.6	2.5	1.1	1.3	2-14															
3:40	2650	60	72	354	244	550	146	51	103	34	63	93	10.5	3.2	1.0	0.8	0.6	2-13	2924	243623	4366	11012	5.9	23.7	1605	0.22	5055	140.7	132.0	143.2		
AVERAGES & TOTALS	2650	502	726	356	243	565	154	49	924	34	62	1566	1174	356	1.5	0.46	1.97	2-12	2921	2148	2055	860	6.4	21.6	1501	1.02	5513	144.8	134.6	146.5		

READINGS NOT CORRECTED

BOBLET &amp; BOUGHTON.

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INSTITUTE OF TECHNOLOGY  
LANSING

MARCH 26, 1909

## LOG OF TEST RUN NO. 5

24 GENERATOR NO. 1

TIME	FUEL USED LBS.	TEMPERATURE DEG. FAHR.											PRESSURES				METER READINGS		GAS ANALYSIS					HEATING VALUE						
		AIR		GAS			WATER						WATER		POWER	PER CENT VOLUME					OF THE GAS									
		OUTSIDE	ROOM	FROM ECONOMIZER	OF GENERATOR OR TO ECONOMIZER	FROM ECONOMIZER	TO SCRUBBER	FROM SCRUBBER	IN FACTORY MAIN	WATER SUPPLY	FROM SCRUBBER	FROM WATER COOLED TOP TO ASH-PIT SEAL	FROM ASH-PIT SEAL	STEAM LBS./SQ. IN.	BOTTOM OF GENERATOR OR IN. OF WATTOI	TOP OF GENERATOR IN. OF WATER	SUCTION OF EXHAUSTER LESS THAT IN THE GENERATOR IN. OF WATER	IN FACTORY MAINS LBS. & OZ.	BAROMETER	INS. OF H <sub>2</sub>	TOTAL WATER CU. FT.	SCRUBBER WATER CU. FT.	WATT METER 10 K W. HR.	CO <sub>2</sub>	CO	H <sub>2</sub>	CH <sub>4</sub>	H <sub>2</sub>	FROM ANALYSIS BTU. PER CU. FT.	FROM ANALYSIS BTU. PER CU. FT. LOW
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31
8:00		50	51	310	225	511	128	51	68	36	62	106	92	4.7	3.8	0.2	3.6	2-9	24587	6668	11593									
8:20		50	53	300	220	518	132	51	90	36	69	111	96	4.7	2.7	0.2	3.5	2-11	28,94			6.5	20.8	19.4	14.5	51.86	1333	141.9	158.3	
8:40		50	53	304	244	500	131	51	90	36	69	110	98	5.0	3.0	0.3	3.6	2-10												
9:00		50	53	308	260	508	131	51	88	36	69	110	101	5.0	2.7	0.8	3.9	2-9	246136	8065	1103									
9:20		51	51	321	260	558	137	51	90	36	76	125	97	4.9	3.0	0.9	3.0	2-10				6.5	18.9	14.4	0.78	59.4	1223	114.1	131.2	
9:40		50	54	318	216	565	136	52	90	36	72	155	100	4.6	1.8	0.4	2.6	2-10												
10:00		51	55	307	222	574	130	52	90	36	72	160	104	4.2	1.1	0.5	2.7	2-9	246380	7175	11976									
10:20		51	58	321	230	530	123	52	86	36	73	156	112	5.5	1.7	0.0	4.4	2-8	2892			5.8	20.2	16.5	0.00	53.5	1330	123.5	144.3	
10:40		52	58	328	242	570	130	53	88	36	84	168	116	5.2	1.6	0.4	3.6	2-9												
11:00		52	56	352	255	640	136	52	88	36	86	182	116	5.2	2.0	1.0	3.3	2-10	246617	7364	11990									
11:20		52	61	357	240	701	134	52	86	36	88	155	112	5.2	2.2	0.0	4.3	2-8	2891			5.8	20.0	17.1	0.76	58.3	135.0	125.9	141.8	
11:40		53	62	356	260	585	138	52	86	36	93	170	114	4.2	2.0	0.0	3.0	2-10												
12:00		54	65	352	284	546	252	54	99	36	93	190	115	3.0	2.8	0.2	2.0	2-10	246866	7587	12008									
12:20		55	69	356	258	594	138	57	92	36	93	192	114	3.3	3.0	0.2	1.3	2-12	2879			5.9	19.9	12.8	1.08	61.9	120.4	113.6	147.9	
12:40		57	69	358	256	596	136	57	96	36	92	190	111	3.8	3.2	0.4	1.6	2-10	247056											
1:00		56	64	361	255	692	136	57	96	36	92	192	102	5.8	3.6	1.0	3.0	2-5	247056	7792	12012									
1:20		57	67	360	260	646	140	57	90	36	90	193	121	3.6	2.0	0.2	2.1	2-6	2879			4.4	21.7	12.8	0.74	60.4	125.8	116.4	135.6	
1:40		57	66	362	272	630	136	57	84	36	88	190	121	4.1	1.8	0.3	2.6	2-5												
2:00		58	70	363	289	610	130	57	95	36	94	187	120	4.4	3.4	0.0	2.4	2-4	287246	7974	12028									
2:20		60	67	378	290	620	138	57	95	36	94	190	122	4.8	2.8	0.0	3.0	2-6	2879			5.3	20.2	14.9	0.75	59.9	128.5	120.1	131.4	
2:40		60	66	360	288	616	136	57	94	36	93	186	124	4.8	2.2	0.0	3.0	2-6												
3:00		61	70	384	290	588	138	57	96	36	93	180	126	4.5	1.9	0.2	3.1	2-5	247634	8151	12048									
3:20		59	66	378	286	623	138	57	96	36	96	185	124	4.5	2.3	0.6	3.0	2-6	2878			5.2	20.1	14.7	1.47	60.5	135.1	126.6	134.6	
3:40		60	66	378	300	618	140	57	97	36	96	190	120	4.6	2.6	0.7	2.8	2-6												
4:00	3125	60	65	380	285	598	142	57	97	36	93	183	118	4.9	3.0	0.2	3.0	2-6	247637	8340	12062									
AVERAGE	3125	54.6	60	346	255	590	134.4	54.3	92.2	36	85.4	123	112	4.6	2.5	0.35	2.98	2-8	28.86	167.6	167.6	1100	58.3	20.2	15.33	0.93	58.08	131.8	123.0	140.0

READINGS NOT CORRECTED.

BOULET &amp; FLIGHTON

ARMOUR  
INSTITUTE OF TECHNOLOGY  
LIBRARY

ARMOUR  
INSTITUTE OF TECHNOLOGY  
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AVERAGE  
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ARMOUR  
INSTITUTE OF TECHNOLOGY  
PURDUE



ARMOUR  
INSTITUTE OF TECHNOLOGY  
LIBRARY

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ARMOUR  
INSTITUTE OF TECHNOLOGY  
BOSTON, MASS.

AVERAGE  
TOTAL

ARMOUR  
INSTITUTE OF TECHNOLOGY  
LIBRARY

APRIL 1, 1909.

LOG OF TEST RUN NO.7

ON GENERATOR NO.2

TIME	FUELED LBS.	TEMPERATURES DEGREE FAHR														PRESSURES				METER READINGS			GAS ANALYSIS					HEATING VALUE				
		AIR						GAS				WATER				STEAM		WATER		POWER		PER CENT BY VOLUME					OF THE GAS.					
		OUTSIDE	ROOM	FROM ECONOMIZER	OF GENERATOR OR TO ECONOMIZER	FROM ECONOMIZER	TO SCRUBBER	FROM SCRUBBER	IN FACTORY MAIN	WATER SUPPLY	FROM SCRUBBER	FROM WATER COOLED TOP TO ASR	FROM ASHPIT SEAL	STEAM LBS./SQ. IN.	BOTTOM OF GENERATOR OR IN WATER	SUCTION OF EXHAUSTER LESS THAT IN GENERATOR INCHES OF WATER	IN FACTORY MAINS LBS. & OZ.	BAROMETER INS. HG.	TOTAL WATER SUPPLY CU. FT.	SCRUBBER WATER CU. FT.	WATT METER 10 KW. H.P.	CO.	CO	H <sub>2</sub>	CH <sub>4</sub>	H <sub>2</sub>	FROM ANALYSIS BTU. PER CU FT. HIGH	FROM ANALYSIS BTU. PER CU FT. LOW	FROM CALORIMETER BTU. PER CU FT. HIGH			
1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	16	17	18	19	20	21	22	23	24	25	26	27	28	29	30	31		
7:30		47	61	320	220	534	138	49	85	38	78	118	92	48	150	0.0	300	2-10	249028	9501	12754											
7:50		44	61	343	245	592	134	48	88	38	74	164	98	50	300	0.3	300	2-9														
8:10		50	64	346	246	600	136	48	88	38	74	154	90	47	160	0.2	3.10	2-10														
8:30		50	65	353	255	644	137	48	90	38	74	70	87	47	140	0.0	3.20	2-9	249187	9642	12768	580	224	1675	156	5349	1511	1410				
8:50		50	66	382	220	644	142	48	90	38	82	137	92	45	190	0.0	3.10	2-7	2954												149.4	
9:10		50	67	380	236	638	141	48	88	38	86	142	96	50	220	0.0	3.80	2-7														
9:30		50	69	381	253	640	140	48	86	38	89	168	104	57	260	0.0	4.40	2-8	249326	9763	12781											
9:50		52	73	394	29.4	674	143	48	87	38	91	167	108	70	180	0.0	4.80	2-8														
10:10		51	73	394	274	644	144	49	87	38	90	160	106	60	160	0.0	4.20	2-8														
10:30		51	73	400	265	654	146	49	87	38	91	143	104	53	200	0.1	3.40	2-8	249463	9888	12975	540	222	149	216	534	1543	1402				
10:50		55	76	409	283	668	146	49	85	38	101	105	113	52	180	1.0	2.80	2-8	2953													157.2
11:10		55	75	406	280	660	144	50	85	38	102	120	110	50	240	0.9	3.60	2-9														
11:30		55	75	400	274	650	142	50	85	38	103	135	103	50	300	0.0	3.90	2-8	249600	10000	12809	640	210	1786	145	534	1486	1370				
11:50		53	80	410	245	600	292	50	90	38	81	84	105	10	180	0.0	0.90	2-8	2953													133.0
12:10		51	76	400	242	580	310	50	90	38	80	80	96	16	140	0.0	1.30	2-9														
12:30		46	73	372	222	506	335	51	89	18	57	76	92	28	000	0.0	1.50	2-10	249813	10198	12822	780	208	1428	2.14	5316	1425	1330				
12:50		46	72	378	230	592	141	51	104	38	83	143	100	45	120	0.1	3.00	2-9	2953													135.3
1:10		48	72	376	244	620	140	51	102	38	66	140	110	44	150	0.2	3.10	2-8														
1:30		48	72	372	270	633	140	52	101	38	71	174	111	42	180	0.0	3.30	2-9	249998	10365	12836	695	212	1711	143	5337	1460	1368				
1:50		48	75	400	275	634	141	52	100	38	70	153	111	45	120	0.0	2.70	2-9	2950													155.2
2:10		46	76	410	280	642	141	52	100	38	72	140	110	40	120	0.0	2.80	2-10														
2:30		45	76	412	290	650	142	52	99	38	74	120	109	35	120	0.0	2.20	2-11	250273	10532	12849	690	208	1444	144	564	1351	1262				
2:50		46	78	407	280	658	138	52	100	38	72	180	117	55	110	0.0	3.30	2-10	2949													144.8
3:10		45	78	412	286	610	144	52	99	38	76	186	116	46	080	0.1	2.80	2-9														
3:30	2375	45	78	418	280	627	148	52	98	38	82	186	120	44	030	0.2	2.60	2-9	250329	10770	12863											
AVERAGES		3875	49	72	387	260	162	50	82	35	80	138	104	45	160	0.1	3.02	2-67	2954	1301	1169	1090	648	2137	1589	170	564	1458	1359	1448		

READINGS NOT CORRECTED

BOBLET &amp; BOUGHTON.

ARMOUR  
INSTITUTE OF TECHNOLOGY  
BIRMINGHAM

TIME

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ARMOUR  
ARCHITECTURE OF TECHNOLOGY  
APPENDIX

VERGES  
OTALS  
RE

ARMOUR  
INSTITUTE OF TECHNOLOGY  
DELRAND



ARMOUR  
INSTITUTE OF TECHNOLOGY  
LIBRARY

## Test Of A 400 H.P. Gas Producer.

### Part 4.

A brief discussion of the various items which enter into the cost of generating gas in a gas producer, and the total cost of this particular gas.



### Test Of A 400 H.P. Gas Producer.

Cost of installation and of operation:-

Regardless of what are the commercial, the constructive principles, and the modes of action of a gas producer, the complete problem of adjusting the conditions so far as possible, of any plant must take into account; (1) cost of fuel to obtain the heat generation along with other accompanying expenses; (2) cost of bulk and weight of the apparatus; (3) profitableness of the investment.

The cost of maintaining and operating a gas producer consists of the following items:

- (a) Interest on the capital invested.
- (b) Depreciation of the plant and buildings.
- (c) Insurance and taxes.
- (d) Fuel cost.
- (e) Cost of cooling water.
- (f) Attendance.
- (g) Maintenance and repairs.

The first three items may be termed "Fixed Charges", and the rest "Operating Charges". The sum of the two is termed the "Total Operating Costs". The fixed charges cover all expenses



### Test Of A 400 H.P. Gas Producer.

which remain constant, whatever the output of the plant may be. The operating charges are those which are proportional to the output of the plant, and which stops when the plant is shut down.

(a) Interest on the investment:- The usual rate of interest on capital in this country is 6 % per annum.

(b) Depreciation of plant and buildings:- The wear and tear on a gas producer installation is very slight. The generators, if reasonably well cared for need be relined only once in two or three years. If not well cared for they may need be relined in three to five months. It is considered in gas producer practice that money will be saved by putting in a good grade of lining when the producer is installed, because the loss incurred in shutting down the producer and putting in a new lining is much greater than the difference in cost between a good and a poor grade of lining. The scrubber should be refilled with coke and excelsior, probably once in a year. The upkeep of the exhauster and the motor is also very small. Taking it altogether, an allowance of



### Test Of A 400 H.P. Gas Producer.

7 % of the investment should cover the depreciation of both the producer and the building. The depreciation on the building should be very small, not more than 2 to 3% of the cost of the building. By making a charge of 7% against the total plant, rental may be included in this item.

(c) Insurance and taxes:- This item is very small since the plant is fireproof throughout. The maximum charge for this item should not exceed 2% of the total investment.

(d) Fuel costs:- This item is most generally the criterion of the economic status of a plant, but in most cases this is not true. Generally, as in this case, the fuel charge is the major part of the expense charge to the plant. But where the plant is of high efficiency, as is the cases with most producer plants, the incidental expenses determine pretty much, whether or not the plant is a paying investment.

The fuel cost varies with the quality of the fuel and the character of the load. If the plant is operating intermittantly the fuel cost will be higher than if operating on the twenty four



## Test Of A 400 H.P. Gas Producer.

hour basis. The coal used in these tests cost \$5.50 per ton delivered on the side track, or \$5.56 per ton unloaded. This coal is an ordinary quality of pea anthracite, usually quite dry. The steam is received from the central station and the cost of producing it is about .4 cts. per boiler H.P. hour, or .0116 cts. per pound ~~per hour~~.

(e) Cost of water for cooling:- Many estimates of the total operating costs totally neglect the costs of water for cooling, but this may in some cases amount to quite an item. At this plant a pumping charge of 7 cts. per 1000 gallons is assumed, that it may be even less than this.

(f) Attendance:- This item depends upon the size of the plant, the nature of the use of the gas, and the number of labor saving devices installed. In the plant tested, there are three men employed for various times. The tenders get 25. cts. per hour one working eleven hours and the other thirteen hours. The ash wheeler gets 17.5 cts. per hour, and works five hours. This is equivalent to one man working 24 hours for 28.6 cts. per hour.

There seems to be an idea among gas users that



### Test Of A 400 H.P. Gas Producer.

a gas producer installation does not require much attention. Gas producer manufacturers should discourage this idea among gas users because their sales depend on the economical working of the plants, and a producer plant cannot go without attention for any length of time.

(g) Maintenance and repairs:- The expenditures for maintenance and repairs should not exceed 2% of the total investment. This refers to the expense of keeping the plant in running order over and above the cost of attendance. It includes cost of upkeep, replacement and precautionary measures, This latter item includes renewal of working parts, painting of perishable or exposed material, and replacing worn out and defective material.

The cost of the gas made is most conveniently expressed per 1000 cubic feet of standard gas. The coal burned per 1000 cubic feet of gas =  $1000 \div 86.6 = 11.55$  lbs. (from Table 3). The steam used per 1000 cubic feet of gas =  $11.55 \times .65 = 7.51$  lbs. The power used to run the exhauster per 1000 cubic feet of gas = 5.22 ...W.Hrs.



Test Of A 400 H.P. Gas Producer.

Reducing this to boiler H.P.Hrs., the power used per 1000 cubic feet of gas =  $5.22 \times .725 = 3.875$  B.H.P.Hrs.. The total water used in gallons per 1000 cubic feet of gas = 79.5.

The cost of material may thus be summed up;

Cost of coal, @ \$5.56 per ton .....	3.176 cts.
Cost of steam, @ .4 per B.H.P.Hr. ....	0.087 cts.
Cost of power, @ .4 cts. per B.H.P.Hr. ...	1.514 cts.
Cost of water, @ 7.0 cts. per 1000 gals...	0.556 cts.
Total cost of material, .....	5.333 cts.

The cost of labor per hour is 28.6 cts. and an average of 24,730 cu.ft. of gas are generated per hour so that the cost for labor per 1000 cu.ft. of gas =  $28.6 \div 24.73 = 1.157$  cts.

Summing up the total fixed charges;

Interest on the investment, .....	6%.
Depreciation of the plant and buildings, ...	7%.
Insurance and taxes, .....	2%.
Maintenance and repairs, .....	3%.
Total charges, .....	18%.

The total investment, including all piping thru the factory, and all the work done on the plant installing it is \$17,563.00. Therefore



Test Of A 400 H.P. Gas Producer.

the total fixed charges =  $\$17,563 \times .18 = \$3161.00$   
 per year, or  $\$3161.00 \div 8760 = 36.1$  cts per hour,  
 or  $36.1 \div 24.73 = 1.419$  cts. per 1000 cu.ft. of  
 gas.

Therefore the total cost of the gas is;

Total cost of material supplied, .....	5.333 cts.
Cost of attendance, .....	1.157 cts.
Fixed charges, .....	1.419 cts.
Total cost per 1000 cu.ft. of gas .....	7.909 cts.

The plant was running under about one-fourth load as an average of the tests. If the plant was running full load, the cost of attendance and the fixed charges would be only one-fourth as much per 1000 cu.ft. of gas, or the cost of the gas under full load conditions would be  $5.333 \div 4$  ( $2.576 \div 4$ ) = 5.977 cts.

The following table shows in a manner, the distribution of the total cost of the plant;

Total charge to the plant, .....	\$17,563.00
Flinn & Dreffein's bill for producer, .....	5,200.00
One 5 H.P. constant speed motor, .....	128.00
One 25 H.P. constant speed motor, .....	400.00
One 25 H.P. variable speed motor, .....	610.00



Test Of A 400 H.P. Gas Producer.

Cost of 2 Root's exhausters, .....	\$958.00
Total cost of apparatus, .....	\$7296.00

Material.

Carpenter's material, .....	\$2898.00
Machinists & gas fitters material, ....	1875.91
Mason's & brick layers material, .....	363.61
Tinsmith's & common labor, .....	75.85
Total cost of material, .....	\$5223.37

Labor.

Carpenter's, .....	\$ 543.83
Machinists & gas fitters, .....	2393.86
Mason's & brick layers, .....	1995.02
Tinsmith's & common labor, .....	110.92
Total cost of labor, .....	\$ 5043.63

The preceding table shows at a glance that there are a great many things charged to the plant that in ordinary producer practice is not charged to the installation. As an illustration of the peculiar charges made against the plant, the writer saw an item on the construction sheet for \$64.60 worth of Garlock packing, for the two exhausters, which is more packing than would be used on these exhausters in at least ten years. There is also



Test Of A 400 H.P. Gas Producer.

another charge of \$46.75 for suppers to the men who worked overtime. From this it may be seen that the total fixed charges which Armour & Co. place against their producer installation is a very broad figure, and could be easily reduced 25 %, and still be a very reasonable figure.



Test Of A 400 H.P. Gas Producer.

Part 5.

Conclusions, and Bibliography.



### Test Of A 400 H.P. Gas Producer.

Conclusions:- There are several noticeable points well worth mentioning as regards this installation.

- (1) Uniform quality of the gas.
- (2) Average high thermal efficiency.
- (3) Flexibility of the installation.
- (4) Simplicity and safety of operation.
- (5) Compactness of the plant.
- (6) Ventilation.
- (7) Rating.

(1) Uniform quality of the gas: The gas generated from this producer never failed to burn in the Junker calorimeter and at the test burners in the gas house. Even at the lightest loads the gas burned with an even blue flame in the calorimeter with a pressure of from .6 to 1.2 inches of water. In the gas house the burners were under a pressure of 2.5 pounds per sq.in., but the gas burned easily and quietly under both conditions. No special burner is required for this gas, as is the case with some kinds of fuel and oil gas. As seen from Table 4, the lowest high heating value of the gas was 128.8 B.T.U., while the highest high heating value was 145.8 B.T.U., with an average of 139.9



### Test Of A 400 H.P. Gas Producer.

B.T.U.. When the gas was running 128.8 B.T.U., only 4.4 lbs. of coal per sq.ft. of grate per hour was being gasified, so that this might be termed rather remarkable.

(2) Average high thermal efficiency: The lowest thermal efficiency found during the test was 78.2 % when the producer was gasifying 4.4 lbs. of coal per sq.ft. of grate per hour, and the highest thermal efficiency, 87.3 % was obtained when gasifying 10.5 lbs. of coal per sq.ft. of grate per hour. The careful attention given to the fuel column, and the pressures in the various parts of the system cannot help but account, in part, for the high average efficiency of 85.6 %. It also seems that the unit is so designed that with ordinary coal, a good average efficiency is sure to result. For example the depth of the water in the ash-pit seal is such that not more than three inches of water pressure can be had at the bottom of the fuel column, thus preventing excessive pressure in the top of the generator. The standby losses are also very small, since the fuel bed is not disturbed when removing the ashes, and



### Test Of A 400 H.P. Gas Producer.

no fuel can be lost in this way.

(3) Flexibility of the installation: The one proof pointing to this fact is shown that at dinner time, from 12:00 to 12:30 the roof valve was not opened while the tests were being made. The steam pressure was lowered (see log of tests, Part 3) and the fires lay waiting to deliver gas when required. Tests of the gas shortly after 12:30 showed it to be of the same quality as that made at any other time. It was stated by one of the tenders that a generator could lie dormant for six days and nights and could be started up in twenty minutes, making as good quality of gas as the one already running. There are no steam boiler installations, or coal fired furnaces of any kind, to the writers' knowledge, which can be brought into full action after being banked, for the same length of time.

(4) Simplicity and safety of operation: Practically unskilled labor is caring for this plant. They are men of ordinary intelligence, without any theoretical training. There are but a few things to do to get the plant started, and these are very



### Test Of A 400 H.P Gas Producer.

simple. The plant is so arranged that the gas will be sucked out of the generator while the ashes and cinders are being removed. The speed of the exhauster may be varied to suit the quantity of gas generated, thus saving power. The steam pressure is easily and quickly regulated to obtain zero gauge pressure in the top of the generator. In the writers' opinion this regulation of the pressure in the top of the generator is one of the best points in the entire installation. For, being thus arranged, no gas can escape to make the room uncomfortable, neither does the air enter to burn the gas. High pressures are entirely absent from the producer, so that no danger can be had from explosions. Various parts of the apparatus are water sealed, making the plant entirely safe.

(5) Compactness of the plant: The plant occupies a floor space of 1188 sq.ft. or 2.97 sq.ft. per rated H.P. This is just a little less than the average space required for producer installations. The arrangement of the plant is such that it may be easily manipulated, and does not require any



### Test Of A 400 H.P. Gas Producer.

unnecessary steps for the tender. An improvement might be made in the arrangement of all gauges on one board, with the steam regulating valve and motor controller at the same point, so that the tender could adjust the steam to bring all gauges to the operating height, without leaving his position.

(6) Ventilation: The ventilation of the producer room is very bad at times; especially when the wind is from the South. Windows are on the North side of the building only, and the wind must be from the West or North to obtain reasonable ventilation. There are four ventilators on the roof, but they do not seem to remove the heat and gases from the lower part of the room. At the present time plans are being drawn for a mechanical draft ventilating system which will no doubt much improve the comfort of the tenders.

(7) Rating: This producer is rated at 400 H.P.. If a gas engine, using this gas had a thermal efficiency of 20 %, the amount of heat required per B.H.P.Hr. would be  $2545 \div .2 = 12725$  B.T.U. which would be approximately 91 cubic feet of this



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gas. The maximum amount of gas generated per hour on one generator was 36738 cubic feet, so that the B.H.P. of one generator would be  $36738 \div 91 = 403$  B.H.P. or 806 B.H.P. for the entire plant. Carrying this load, the producer did not seem to be overloaded, but seemed rather to be working at its best at this rate of operation.

The Messrs. Flinn & Dreffein, consulting engineers, who designed this plant, are graduates of the Armour Institute Of Technology. It would not be more than proper to give these gentlemen the credit of having designed, installed and placed in operation, a model producer gas plant, complete in its details, and a plant which should be studied even closer than has been done in this investigation.

Respectfully submitted,

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